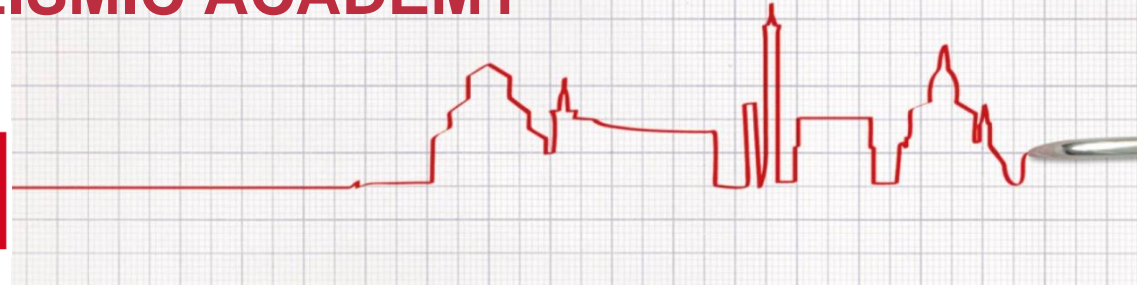
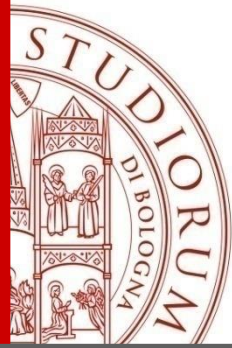


VII EDIZIONE SEISMIC ACADEMY

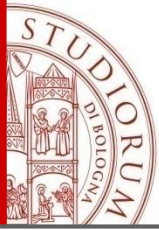
10 Ottobre 2019, Bologna

SEISMIC
ACADEMY



RISPOSTA SISMICA E PROGETTAZIONE SEMPLIFICATA DI STRUTTURE DOTATE DI DISSIPATORI VISCOSI

Stefano Silvestri
Università di Bologna



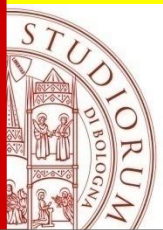
Presentation outline

- Background
- Previous studies
- The “five-step procedure” (2010)
- The “direct five-step procedure” (2016-2018)
- Applicative example
- Conclusions and future developments

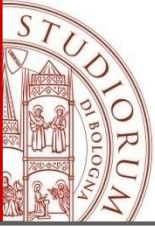
OBJECTIVES

To develop an easy method for preliminary quick design of structures equipped with dampers (for the wide diffusion of their use)

To give fully-analytical tools to the professional engineers for the control of the results of non-linear TH analyses

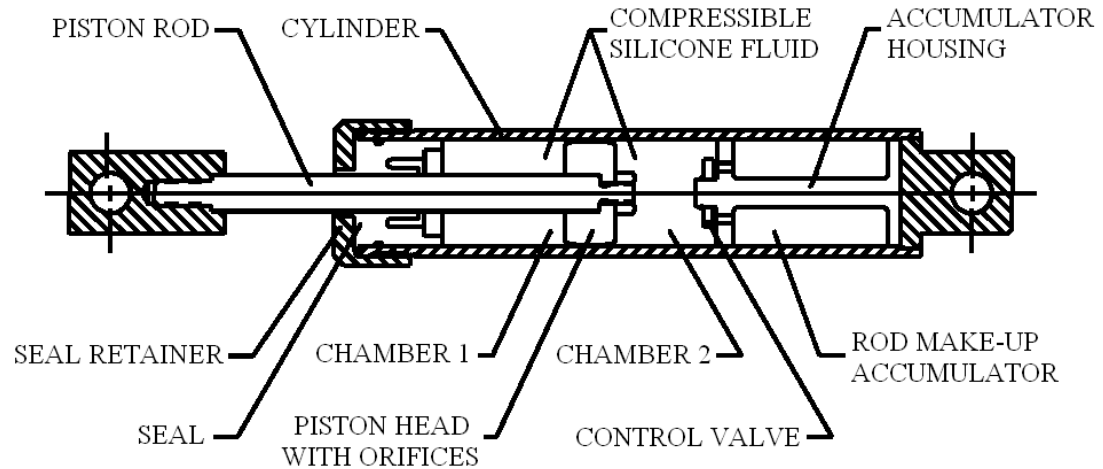


Background

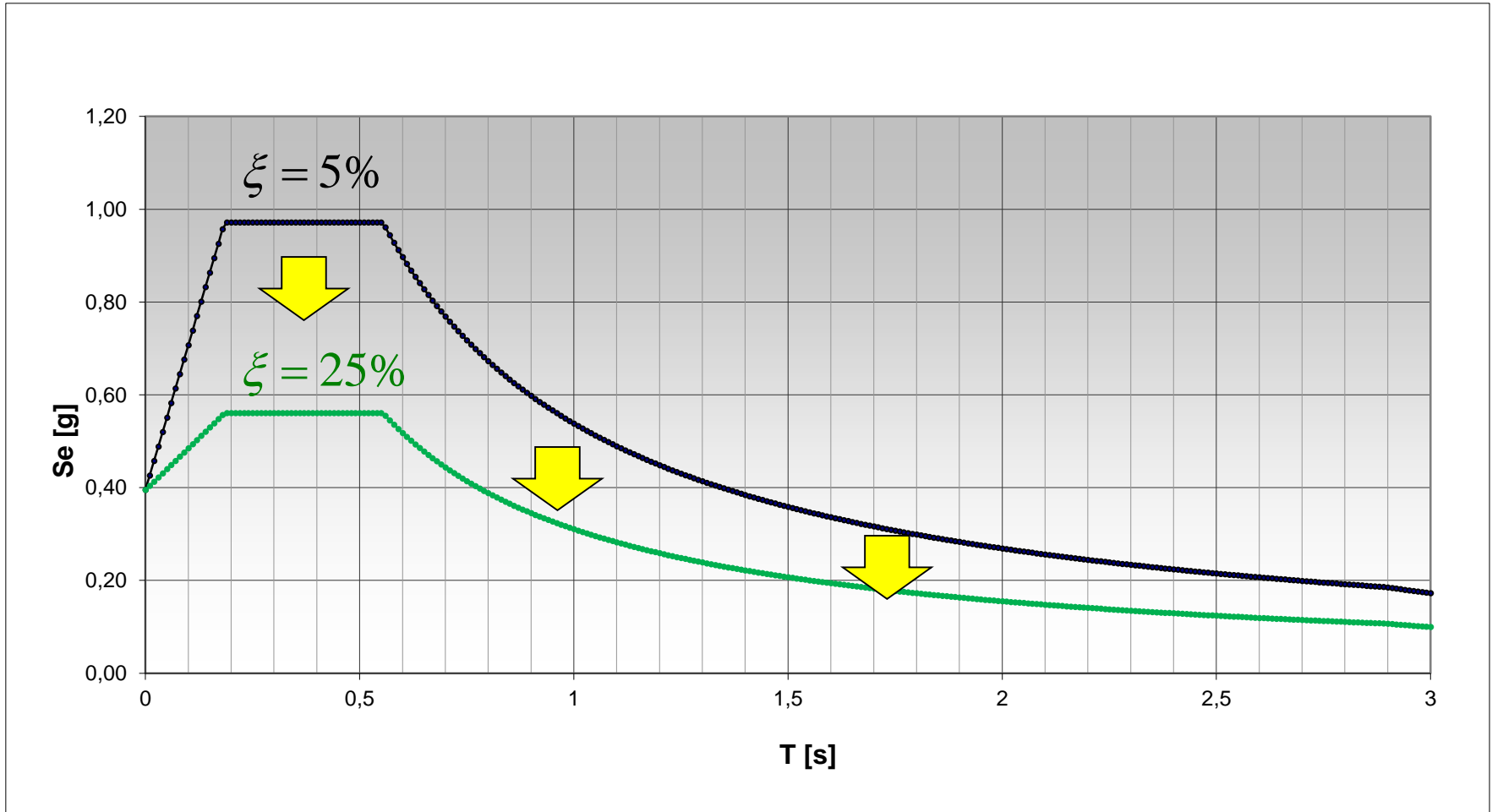


Viscous dampers

energy is dissipated in the guise of **heat** through the passage of a **viscous silicone fluid** (stable w.r.t. temperature) across the piston head with orifices

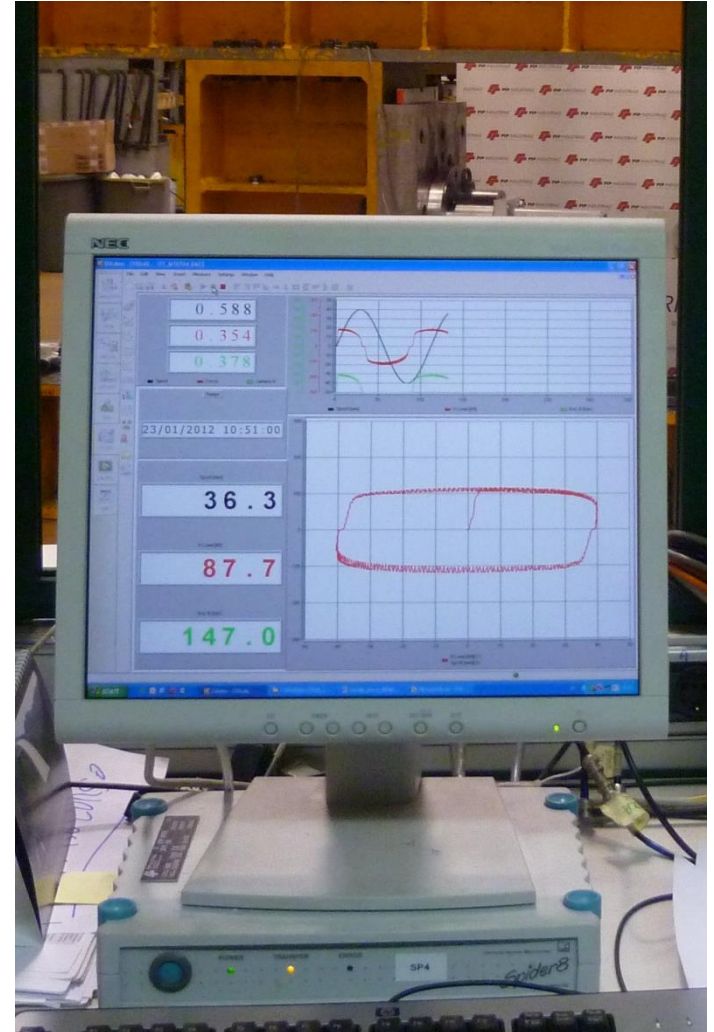


Benefits provided by viscous dampers

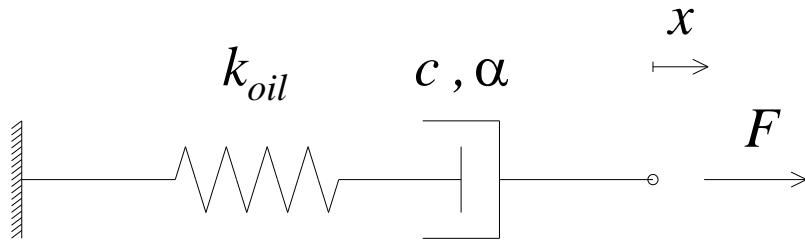


Viscous dampers

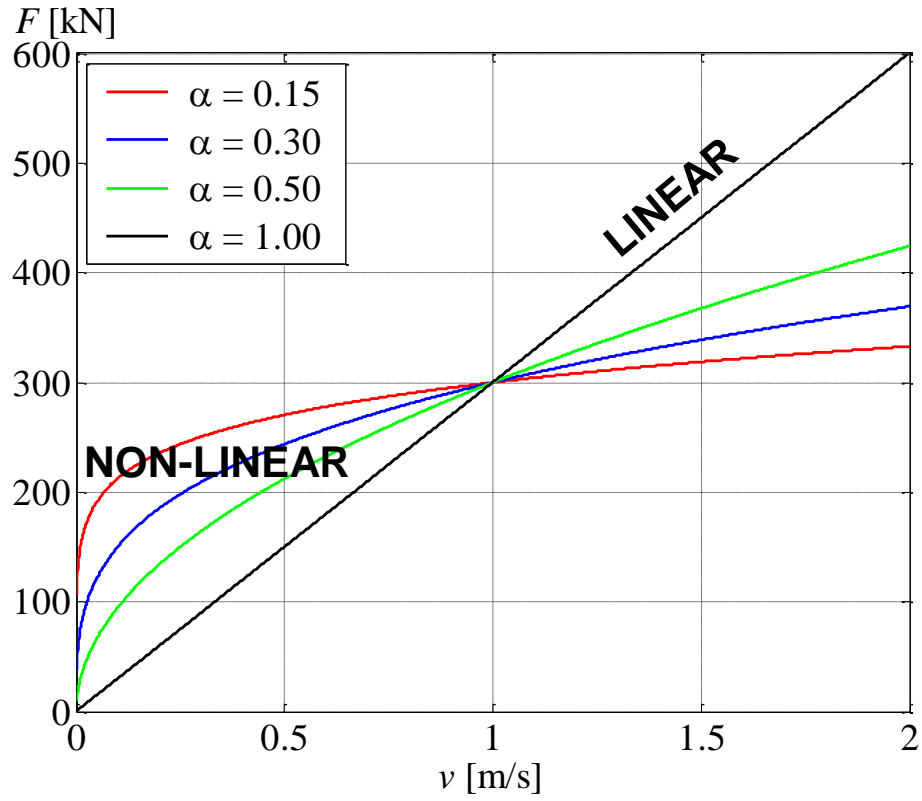
constant velocity tests



Viscous dampers



$$F = k_{oil} \cdot x = c \cdot v^\alpha$$

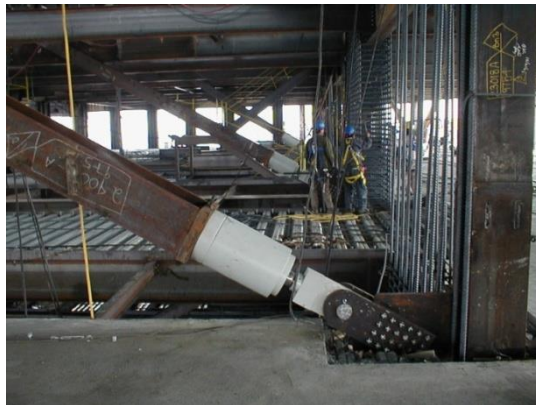
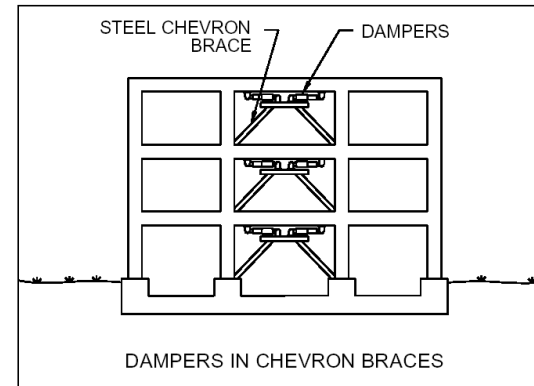
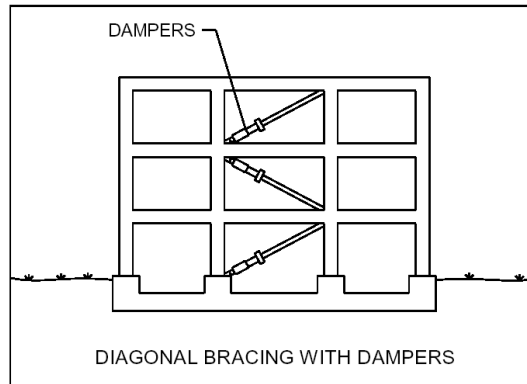


$$\alpha \cong 0.15$$

$$k_{oil} = \frac{F_{max}}{5\% \left(\frac{x_{max}}{2} \right)}$$

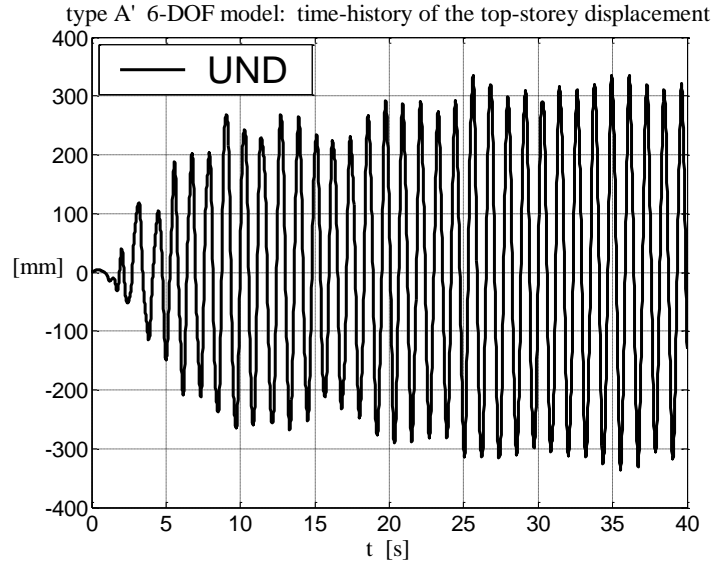
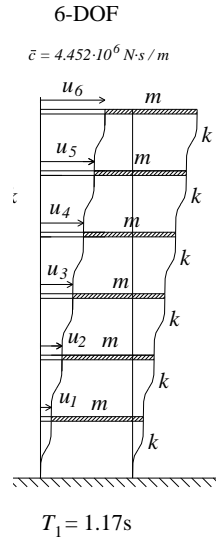
Effectiveness

- The **effectiveness** of fluid viscous dissipative devices in reducing the seismic demand on the structural elements has been demonstrated by a number of research works and real applications since the 1980s.



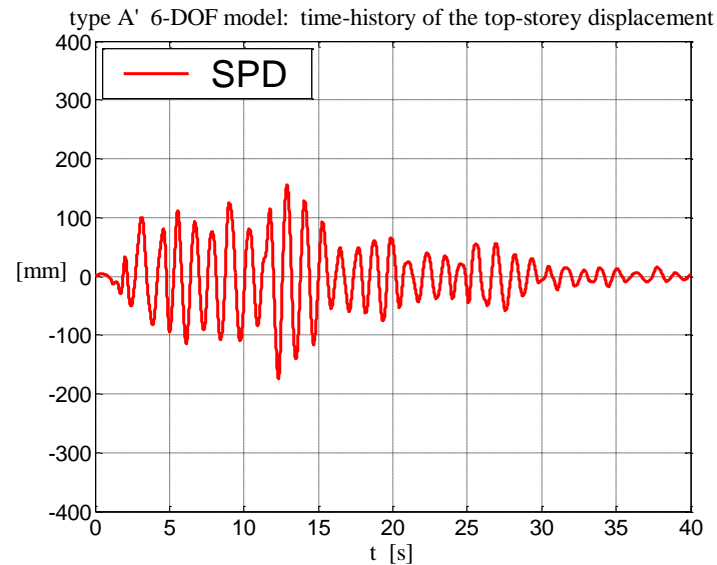
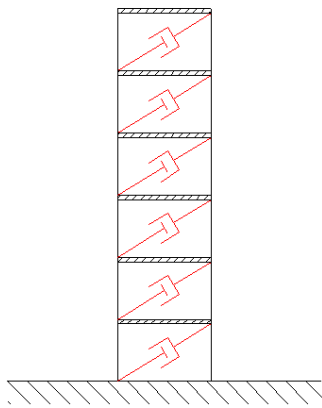
Seismic Dampers installed at the
Hotel Woodland, in Woodland, California
Force = 100,000 lbs., Stroke = +/- 2 inches
Production = 16 pieces

Effectiveness

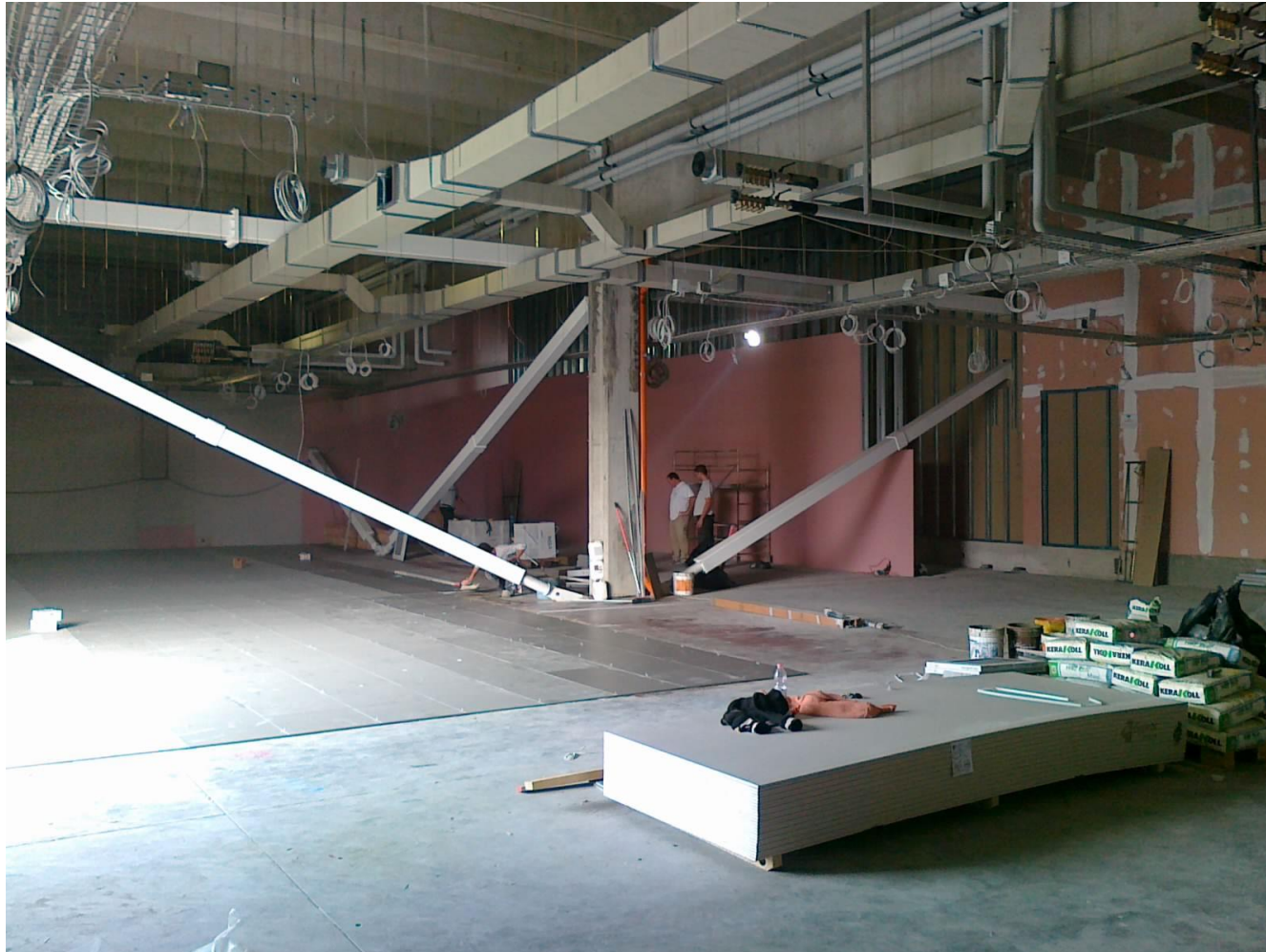


**top-storey
displacement
response**

El Centro 1940
scaled to
PGA = 0.3g



Real application



*pictures
courtesy of
Ing. Franco
Baroni,
Studio Ceccoli
& Associati,
Bologna*

*dampers
manufactured
by FIP,
Padova,
Italy*

Real application



*pictures
courtesy of
Ing. Franco
Baroni,
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& Associati,
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*dampers
manufactured
by FIP,
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Italy*

Real application



*pictures
courtesy of
Ing. Franco
Baroni,
Studio Ceccoli
& Associati,
Bologna*

Real application

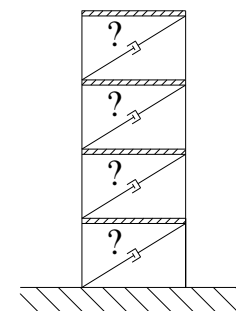
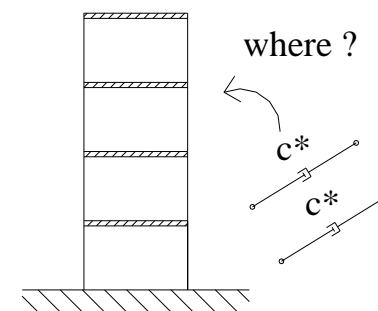
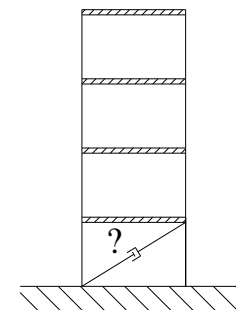


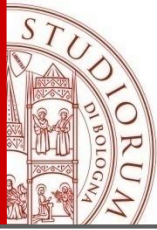
*pictures
courtesy of
Ing. Franco
Baroni,
Studio Ceccoli
& Associati,
Bologna*

In the scientific literature:


- **Sophisticated numerical algorithms for dampers optimization** (Takewaki 1997, 2000 and 2009, Shukla and Datta 1999, Lopez Garcia 2001, Singh and Moreschi 2002, Levy and Lavan 2006, ...)
 - **Computational expertise and time** (beyond the typical availabilities of the designers) are needed
 - **Numerical results** which do not provide physical insight into the matter.
- The issue of developing **simple/analytical methods** in order to size and locate the viscous dampers is still open.

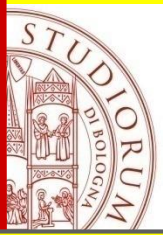
(like equivalent static analysis vs. non-linear time-history analysis)





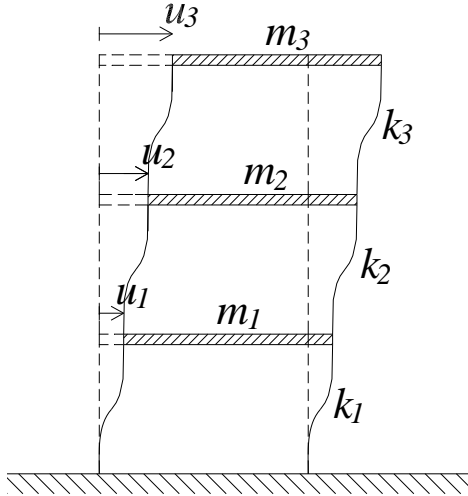
Design procedures

- report NCEER-92-0032 (Constantinou and Symans 1992)
- report MCEER-00-0010 (Ramirez et al. 2000)
- **ASCE 7 (2005) Chapter 18**, which is grounded on the MCEER-00-0010 approach and on the works by Ramirez et al. (2002a and b, and 2003) and by Whittaker et al. (2003), contains systematic procedures for design and analysis of building with damping systems (*use of the residual mode approach*).
- **Lopez-Garcia 2001** developed a simple **algorithm** for optimal damper configuration (placement and properties) in MDOF structures, assuming a constant inter-storey height and a straight-line first modal shape.
- **Christopoulos and Filiatrault (2006)** suggested a design approach for estimating the damping coefficients of added viscous dampers consisting in a *trial and error procedure*.
- **Silvestri et al. 2010**  “five-step procedure”



Previous studies

Insight into the Rayleigh damping



$$\underline{\tilde{m}} \ddot{\underline{u}} + \underline{\tilde{c}} \dot{\underline{u}} + \underline{\tilde{k}} \underline{u} = -\underline{\tilde{m}} \underline{1} \cdot \ddot{u}_g(t)$$

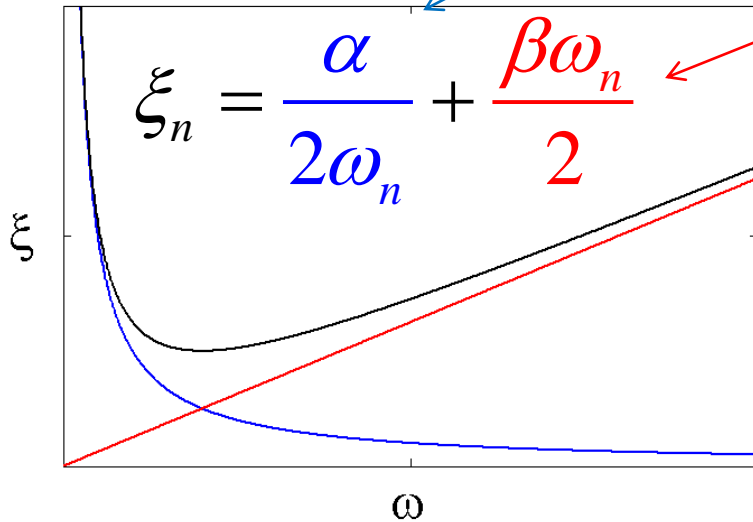
\underline{u} physical coordinates vector

$\underline{\tilde{m}}$ mass matrix

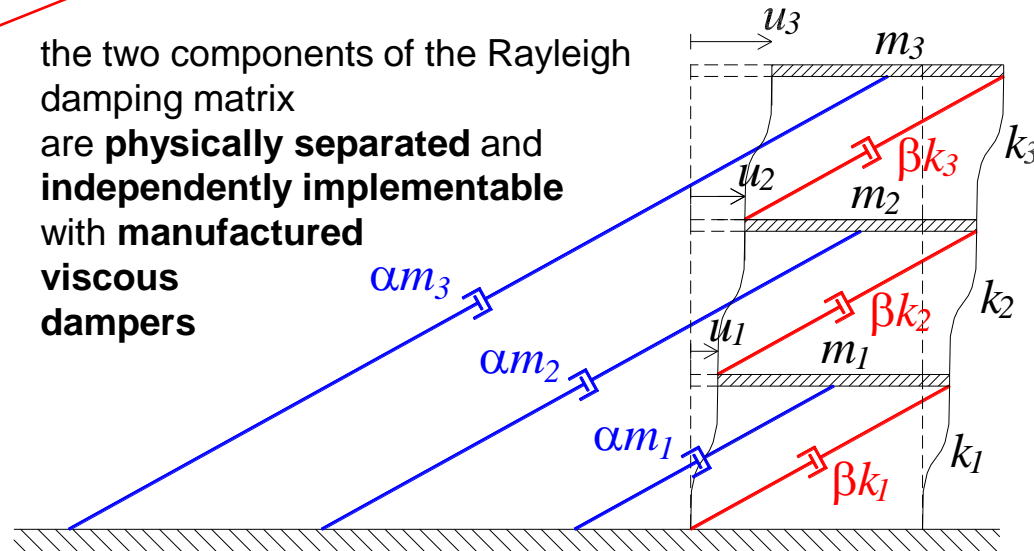
$\underline{\tilde{k}}$ stiffness matrix

$$\underline{\tilde{c}} = \alpha \underline{\tilde{m}} + \beta \underline{\tilde{k}}$$

$$\underline{\tilde{c}} = \alpha \begin{bmatrix} m_1 & 0 & 0 \\ 0 & m_2 & 0 \\ 0 & 0 & m_3 \end{bmatrix} + \beta \begin{bmatrix} k_1 + k_2 & -k_2 & 0 \\ -k_2 & k_2 + k_3 & -k_3 \\ 0 & -k_3 & k_3 \end{bmatrix}$$



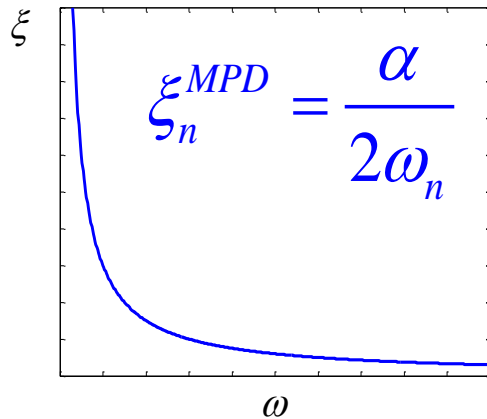
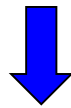
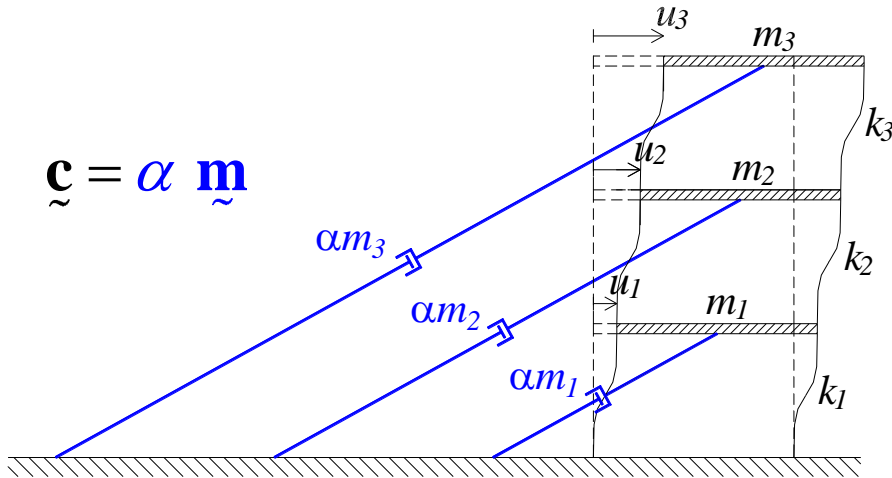
the two components of the Rayleigh damping matrix are **physically separated** and **independently implementable** with **manufactured viscous dampers**



Properties of **MPD** and **SPD** systems (1)

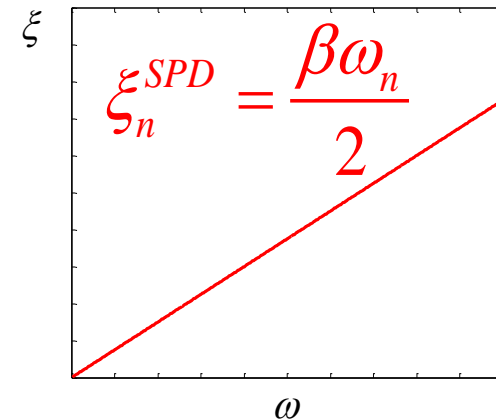
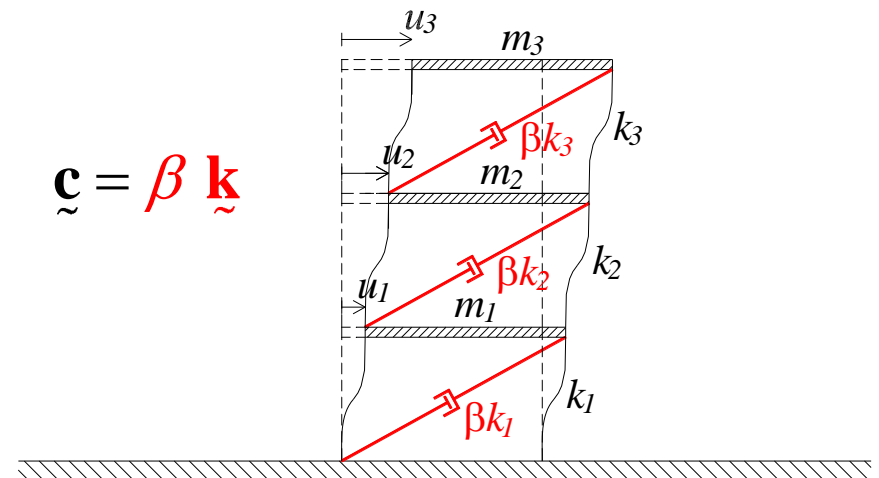
Mass Proportional Damping system

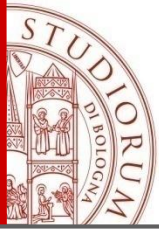
$$\underline{\zeta} = \alpha \underline{m}$$



Stiffness Proportional Damping system

$$\underline{\zeta} = \beta \underline{k}$$



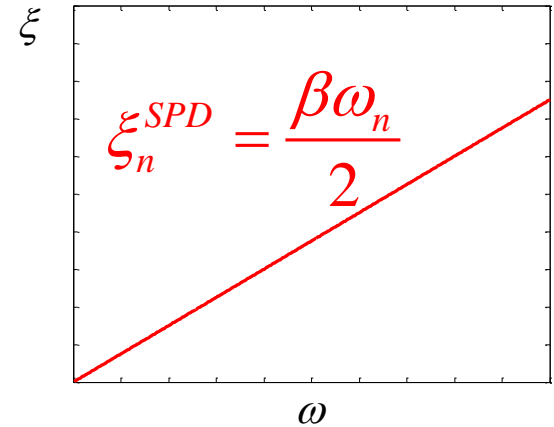
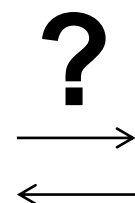
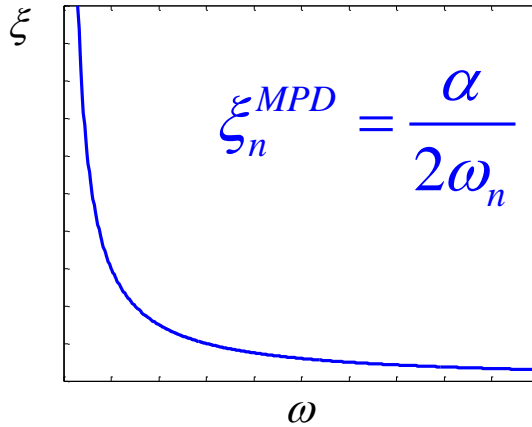


Properties of MPD and SPD systems (2)

HP:

The two systems have equal total damping:

$$c_{tot} = \sum c_i$$

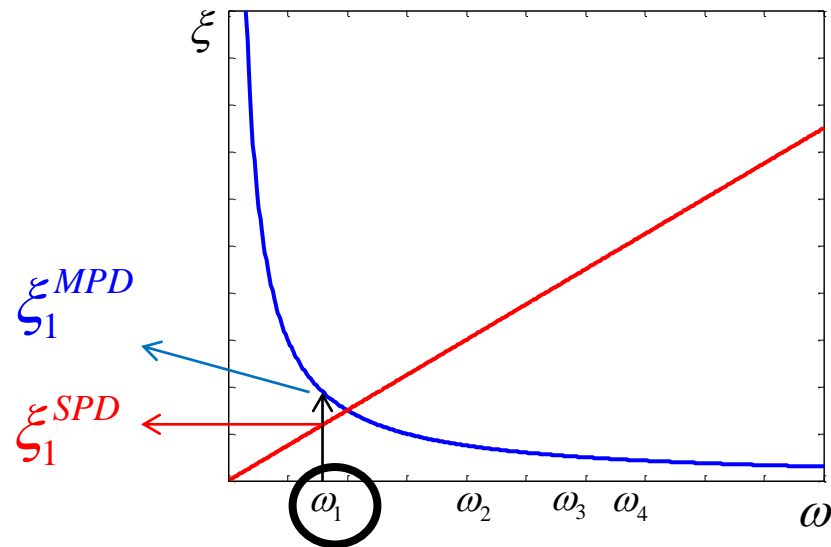


QUESTION:

How the modal damping ratios of the SPD system and the “corresponding” MPD system are related to each other?

?

$$\xi_1^{MPD} \begin{matrix} \geq \\ \leq \end{matrix} \xi_1^{SPD}$$



Properties of MPD and SPD systems (3)

From basic dynamics:

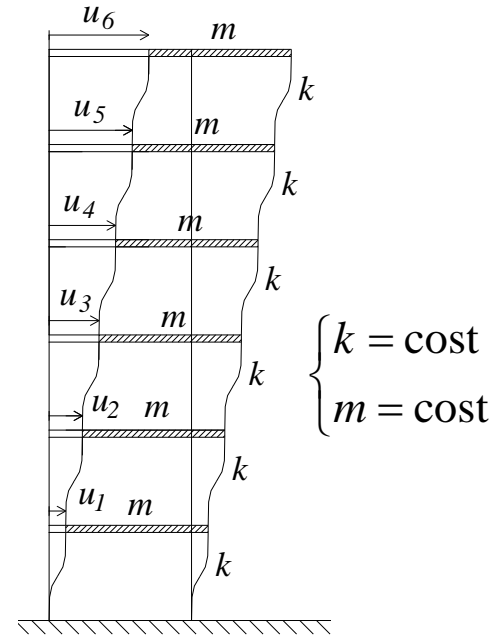
$$\frac{\xi_i^{SPD}}{\xi_i^{MPD}} = \frac{\frac{\beta}{2} \omega_i}{\frac{\alpha}{2\omega_i}} = \frac{\beta}{\alpha} \omega_i^2 = \dots$$



HP1) structures characterised by constant values of lateral stiffness k and storey mass m

HP2) equal "total cost" constraint

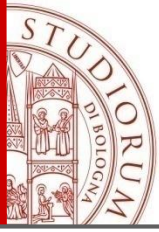
$$\left. \begin{aligned} \bar{\alpha} &= \frac{\bar{c}}{\sum_{i=1}^N m_i} = \frac{\bar{c}}{Nm} \\ \bar{\beta} &= \frac{\bar{c}}{\sum_{i=1}^N k_i} = \frac{\bar{c}}{Nk} \end{aligned} \right\} \bar{\alpha}m = \bar{\beta}k \Rightarrow \frac{\bar{\beta}}{\bar{\alpha}} = \frac{m}{k} = \frac{1}{\omega_0^2} \quad \left(\omega_0 = \sqrt{\frac{k}{m}} \right)$$



$$\frac{\xi_i^{SPD}}{\xi_i^{MPD}} = \frac{\frac{\bar{\beta}}{2} \omega_i}{\frac{\bar{\alpha}}{2\omega_i}} = \frac{\bar{\beta}}{\bar{\alpha}} \omega_i^2 = \frac{\omega_i^2}{\omega_0^2} = \dots$$



$$\frac{\xi_i^{SPD}}{\xi_i^{MPD}} = \frac{\frac{\bar{\beta}}{2} \omega_i}{\frac{\bar{\alpha}}{2\omega_i}} = \frac{\bar{\beta}}{\bar{\alpha}} \omega_i^2 = \frac{\omega_i^2}{\omega_0^2} = \Omega_i^2 = \Lambda_i$$



Properties of **MPD** and **SPD** systems (4)

$$\frac{\xi_i^{SPD}}{\xi_i^{MPD}} = \frac{\frac{\bar{\beta}}{2} \omega_i}{\frac{\bar{\alpha}}{2\omega_i}} = \frac{\bar{\beta}}{\bar{\alpha}} \omega_i^2 = \frac{\omega_i^2}{\omega_0^2} = \Omega_i^2 = \Lambda_i \longrightarrow \text{eigenproblem}$$

The matrix $\mathbf{A}_N = \frac{1}{\omega_0^2} \mathbf{k}_N \mathbf{m}_N^{-1} =$

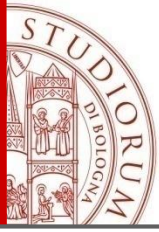
$$\begin{bmatrix} 2 & -1 & 0 & \dots & & 0 \\ -1 & 2 & -1 & 0 & \dots & 0 \\ 0 & -1 & 2 & \dots & \dots & \dots \\ \dots & 0 & \dots & \dots & \dots & \dots \\ & & \dots & \dots & -1 & 0 \\ & & & & -1 & 2 & -1 \\ 0 & \dots & \dots & 0 & -1 & 1 \end{bmatrix}$$

has eigenvalues Λ_i

$$\text{if } \Lambda_i < 1 \Rightarrow \xi_i^{SPD} < \xi_i^{MPD}$$



The i -th damping ratio of the SPD system is less than the “corresponding” one of the MPD system (the two systems with same total damping)



Properties of MPD and SPD systems (5)

► It can be demonstrated:

$$\frac{\xi_1^{SPD}}{\xi_1^{MPD}} = \Lambda_1 < 1$$

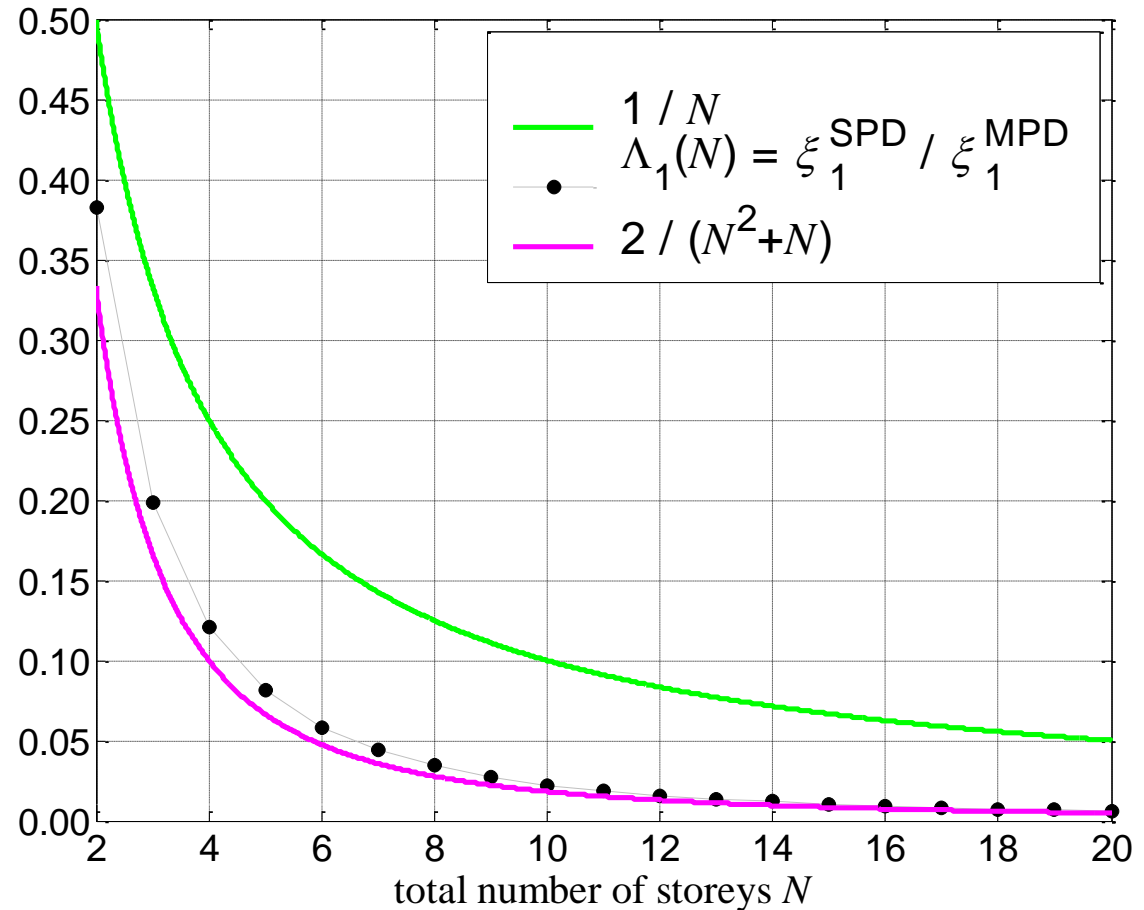
► Upper bound:

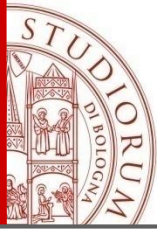
$$\frac{\xi_1^{SPD}}{\xi_1^{MPD}} < \frac{1}{N}$$

► Approximation:

$$\frac{\xi_1^{SPD}}{\xi_1^{MPD}} \cong \frac{2}{N(N+1)}$$

*





Properties of **MPD** and **SPD** systems (6)



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Journal of Sound and Vibration 292 (2006) 21–58

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On the modal damping ratios of shear-type structures equipped with Rayleigh damping systems

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DISTART, Facoltà di Ingegneria, Università degli Studi di Bologna, Viale Risorgimento 2, 40136 Bologna, Italy

Received 29 July 2002; received in revised form 6 July 2005; accepted 15 July 2005

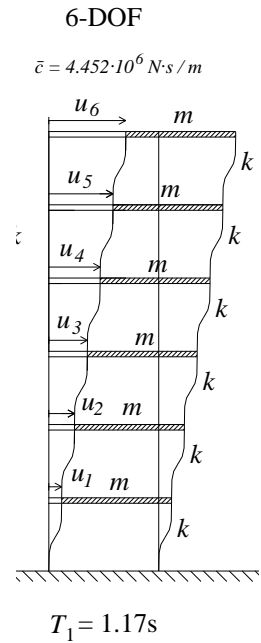
Available online 28 September 2005

Abstract

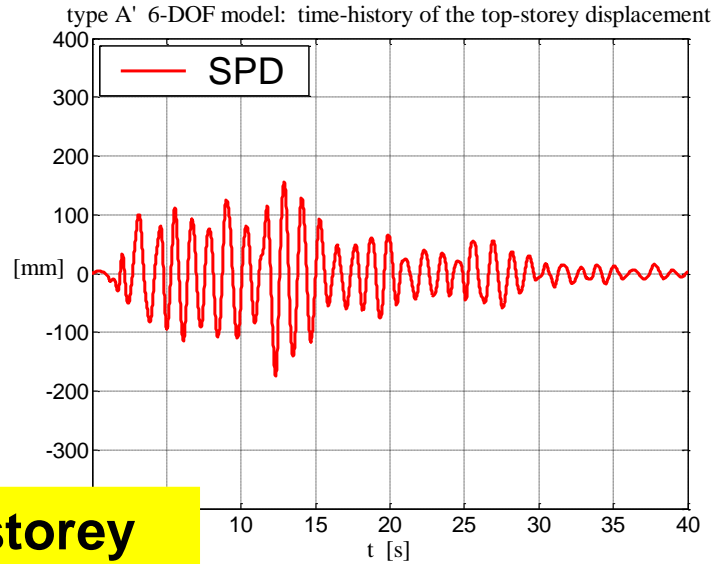
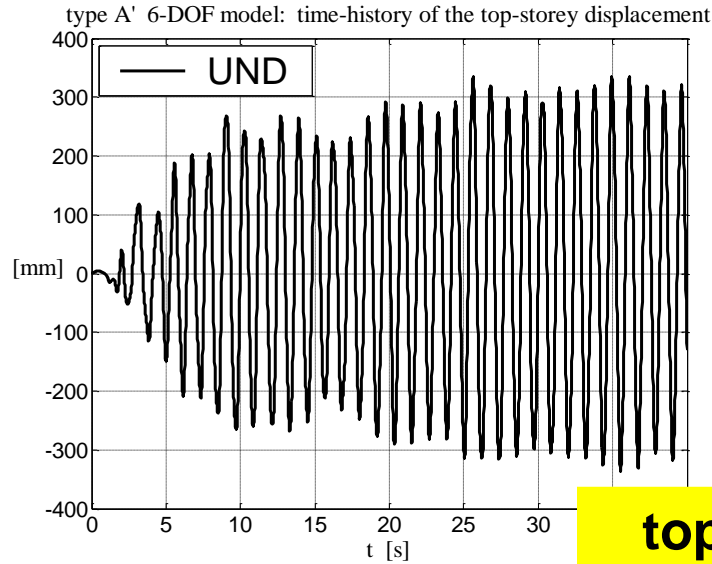
The effects of added manufactured viscous dampers upon shear-type structures are analytically investigated here for the class of Rayleigh damping systems. The definitions of mass proportional damping (MPD) and stiffness proportional damping (SPD) systems are briefly recalled and their physical counterpart is derived. From basic physics, a detailed mathematical demonstration that the first modal damping ratio of a structure equipped with the MPD system is always larger than the first modal damping ratio of a structure equipped with the SPD system is provided here. All results are derived for the class of structures characterised by constant values of lateral stiffness and storey mass, under the equal “total size” constraint. The paper also provides closed form demonstrations of other properties of modal damping



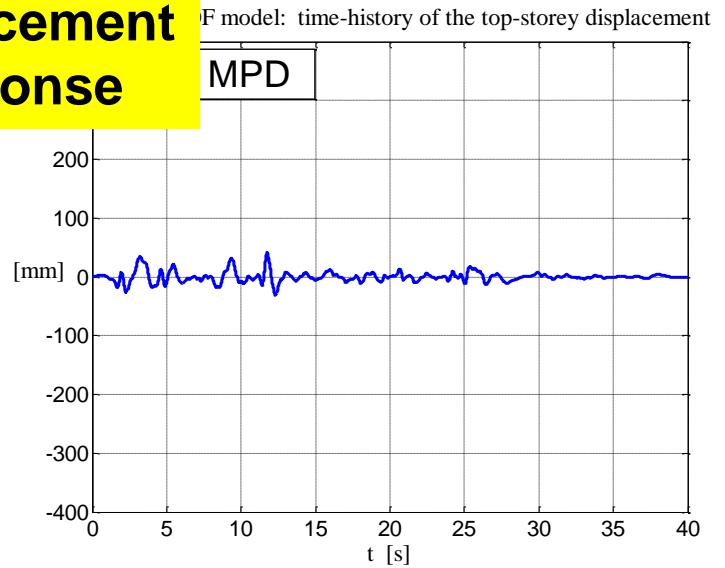
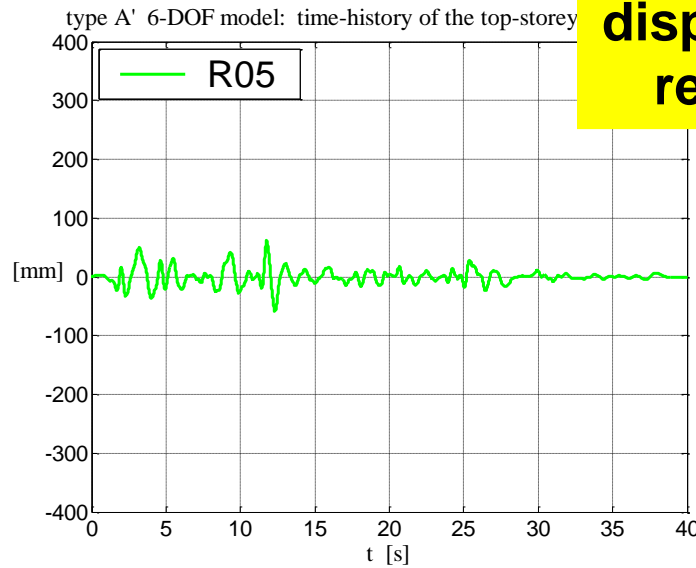
Seismic response of **MPD** and **SPD** systems



El Centro 1940
scaled to
PGA = 0.3g



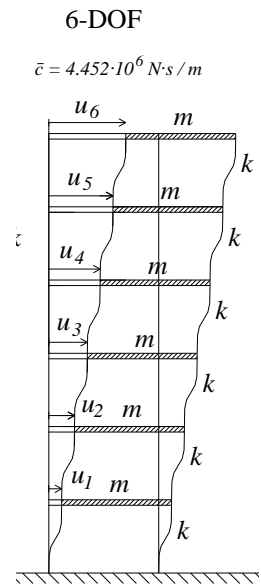
**top-storey
displacement
response**





Seismic response of **MPD** and **SPD** systems

type A' 6-DOF model: averages over 40 earthquake ground motions



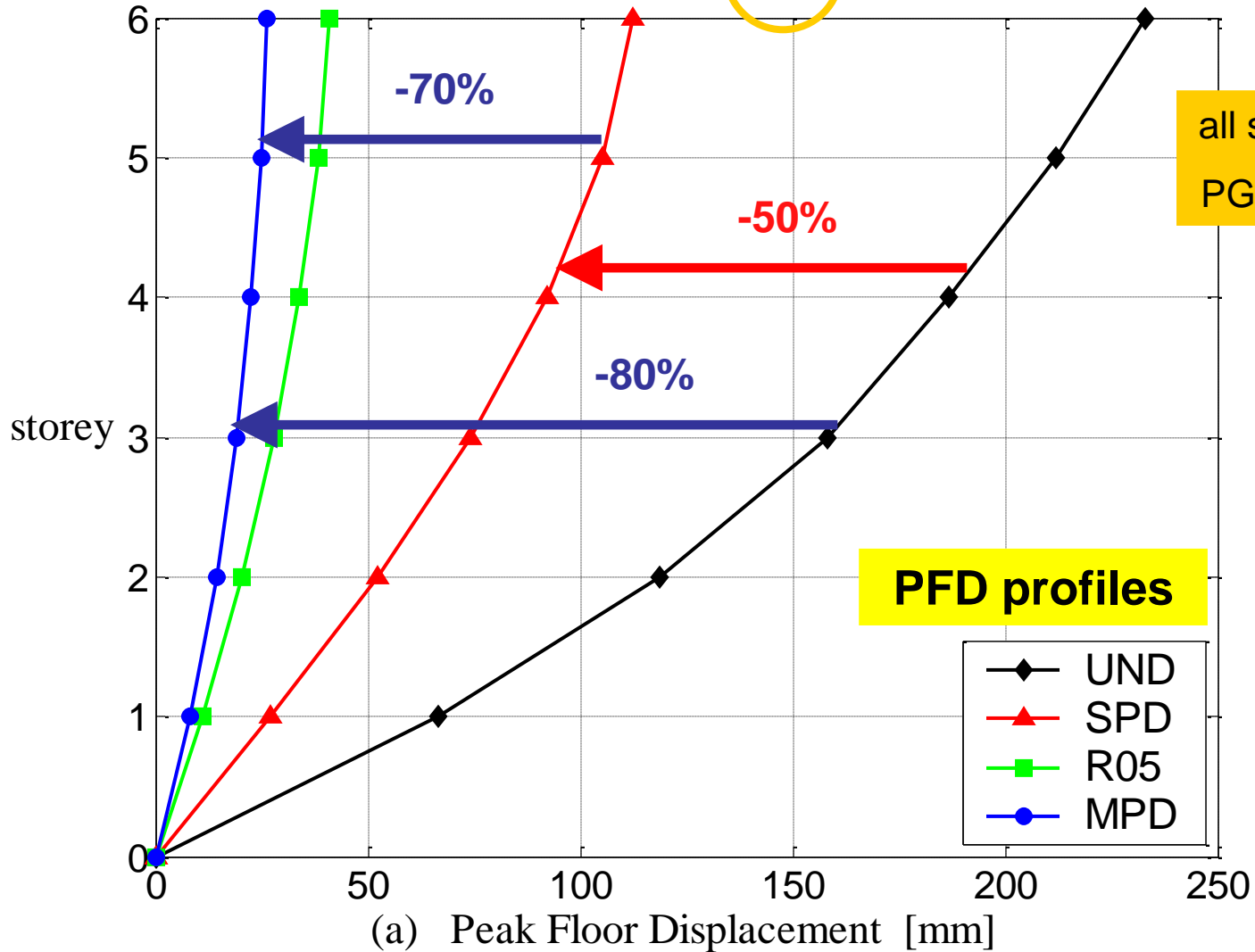
$T_1 = 1.17\text{s}$

$\xi_1^{MPD} = 86\%$

$\xi_1^{SPD} = 5\%$

$\xi_2^{MPD} = 29\%$

$\xi_2^{SPD} = 15\%$



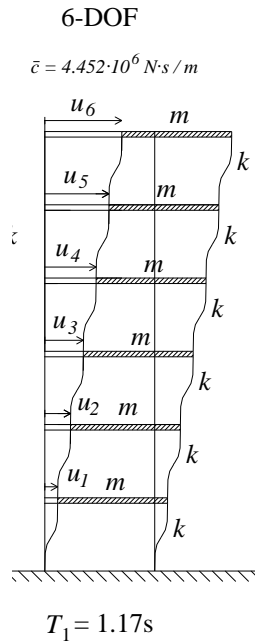
all scaled to
PGA = 0.3g

PFD profiles

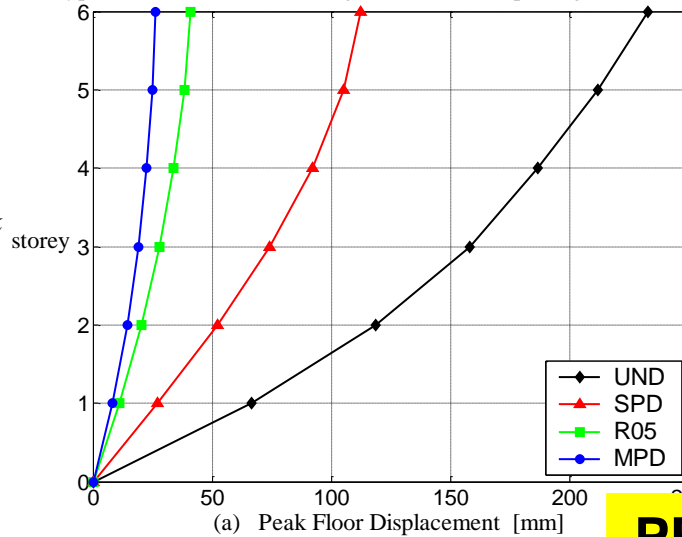
- ◆ UND
- ▲ SPD
- R05
- MPD



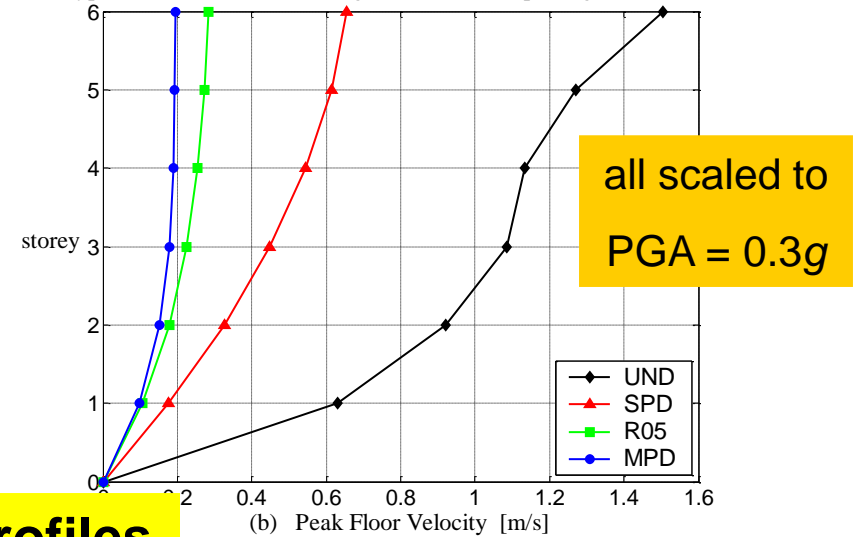
Seismic response of **MPD** and **SPD** systems



type A' 6-DOF model: averages over 40 earthquake ground motions



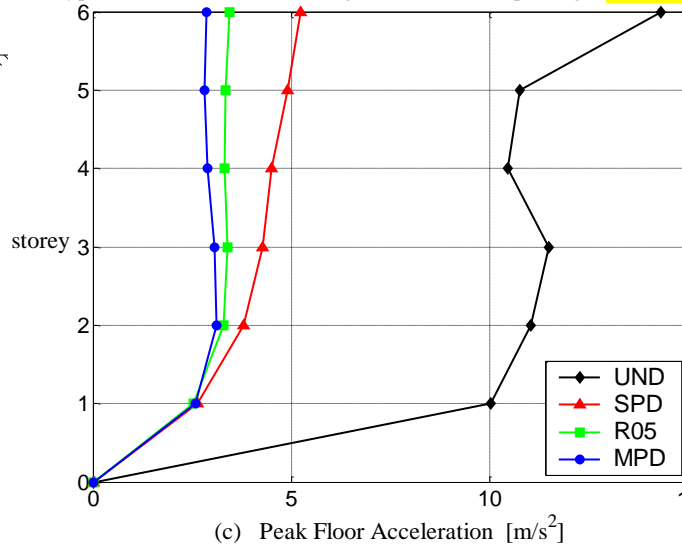
type A' 6-DOF model: averages over 40 earthquake ground motions



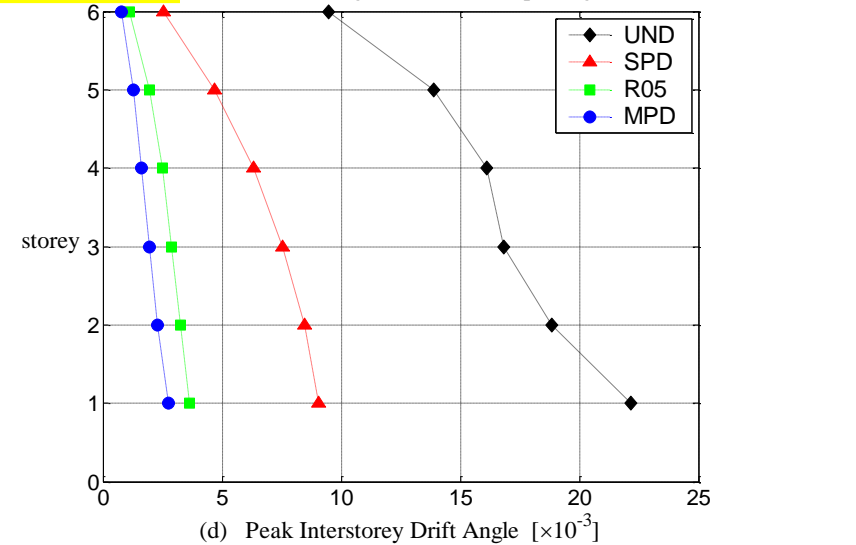
all scaled to
PGA = 0.3g

PF profiles

type A' 6-DOF model: averages over 40 earthquake ground motions

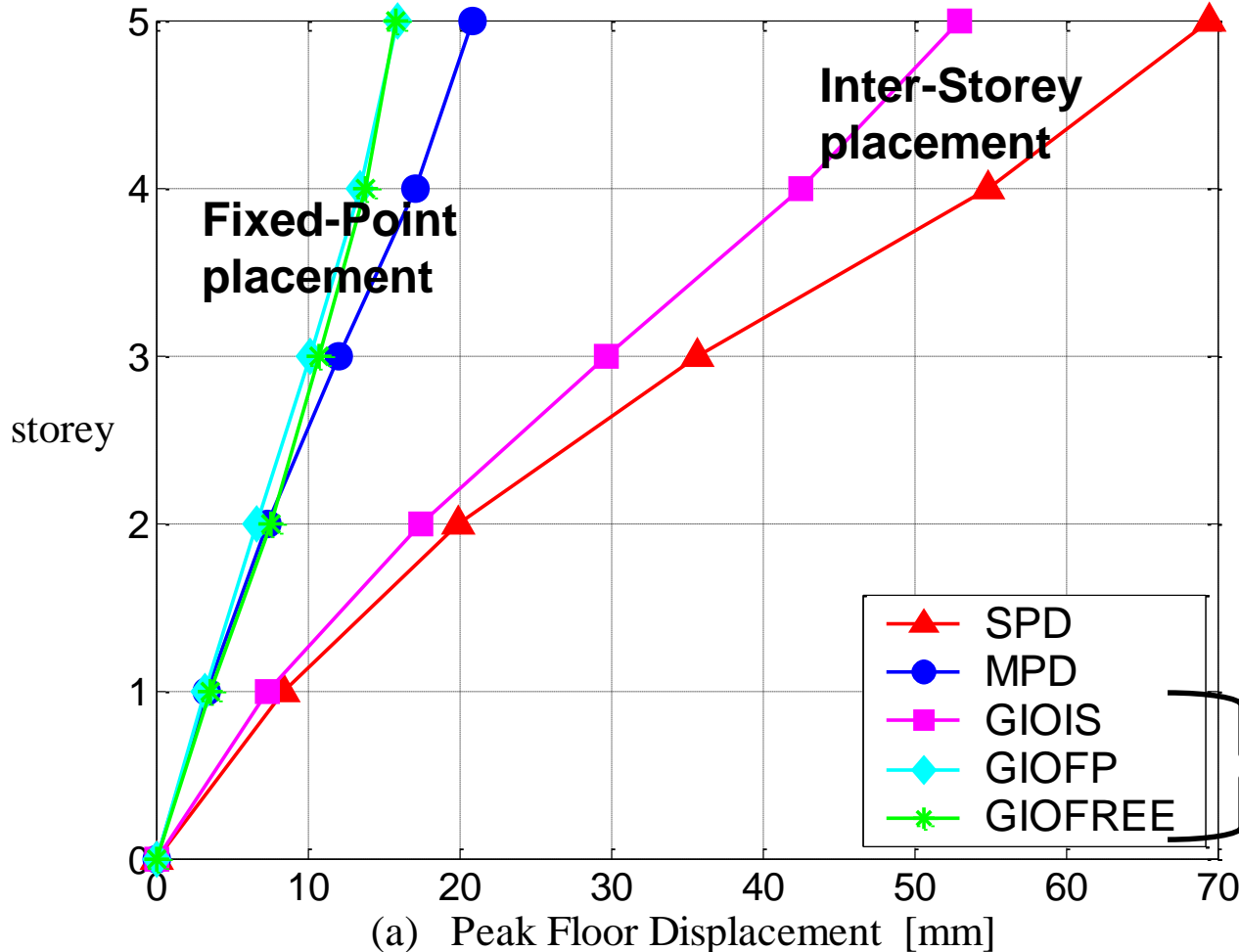


type A' 6-DOF model: averages over 40 earthquake ground motions



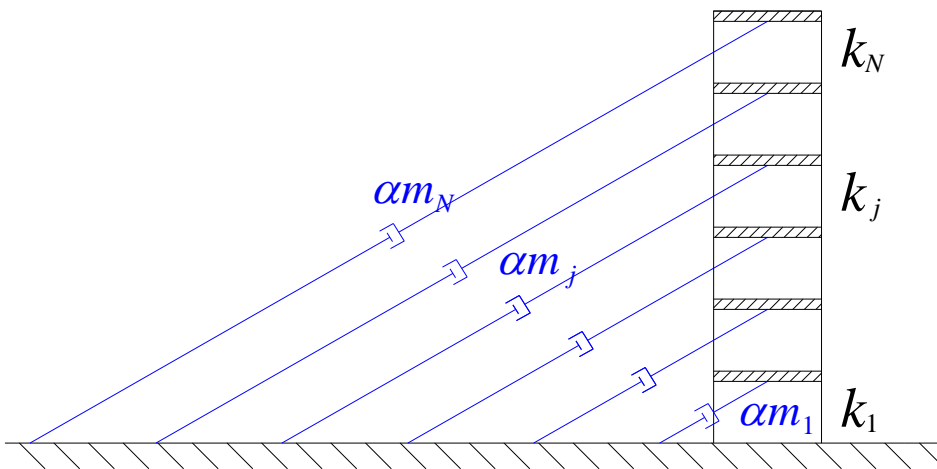
Seismic response of Genetically Identified Optimal (GIO) systems

5-storey r.c. structure: averages over 40 earthquake ground motions



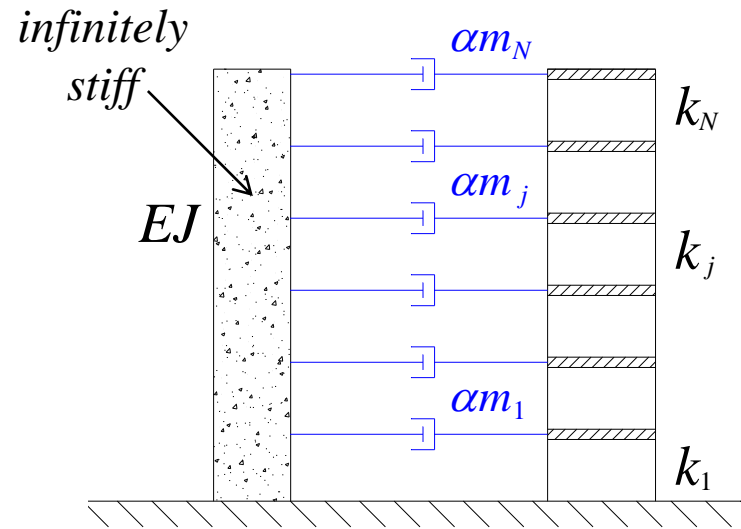
Objective function:
*average of the
standard deviations
of the interstory
drift angles.*

Implementation of **MPD** systems



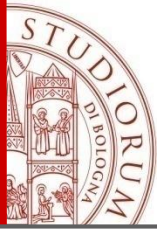
DIRECT IMPLEMENTATION

Use of long buckling-resistant braces
(mega-braces, unbonded braces, prestressed steel cables, ...)



INDIRECT IMPLEMENTATION

Dampers placed between the structure and a very stiff vertical lateral-resistant element



Properties of **MPD** and **SPD** systems (7)

$$\underline{\mathfrak{c}} = \alpha \underline{\mathfrak{m}}$$

$$\xi_1^{MPD} = \frac{\alpha}{2\omega_1} \quad \alpha = 2 \cdot \xi_1^{MPD} \cdot \omega_1$$

$$\underline{\mathfrak{c}} = 2 \cdot \xi_1^{MPD} \cdot \omega_1 \cdot \underline{\mathfrak{m}}$$

$$c_{storey,j} = 2 \cdot \xi_1^{MPD} \cdot \omega_1 \cdot m_j$$

m_j is easy to be calculated

$$c_{tot,MPD} = 2 \cdot \xi_1^{MPD} \cdot \omega_1 \cdot m_{tot}$$

$$\underline{\mathfrak{c}} = \beta \underline{\mathfrak{k}}$$

$$\xi_1^{SPD} = \frac{\beta\omega_1}{2} \quad \beta = \frac{2 \cdot \xi_1^{SPD}}{\omega_1}$$

$$\underline{\mathfrak{c}} = \frac{2 \cdot \xi_1^{SPD}}{\omega_1} \cdot \underline{\mathfrak{k}}$$

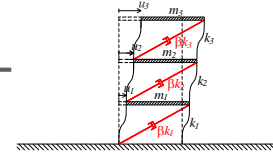
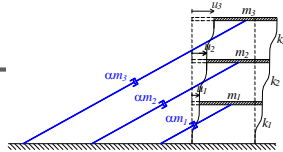
$$c_{storey,j} = \frac{2 \cdot \xi_1^{SPD}}{\omega_1} \cdot k_j$$

k_j is not so immediate ...

$$c_{tot,SPD} = \dots$$



Properties of **MPD** and **SPD** systems (8)



$$C_{tot,MPD} = 2 \cdot \xi_1^{MPD} \cdot \omega_1 \cdot m_{tot}$$

$$C_{tot,SPD} = 2 \cdot \xi_1^{SPD} \cdot \omega_1 \cdot m_{tot} \cdot \frac{N(N+1)}{2}$$

$$\xi_1^{MPD} = \frac{C_{tot}}{2 \cdot \omega_1 \cdot m_{tot}}$$

$$\xi_1^{SPD} = \frac{C_{tot}}{2 \cdot \omega_1 \cdot m_{tot}} \cdot \frac{2}{N(N+1)}$$

$$\xi_1^{SPD} \frac{N(N+1)}{2} = \frac{C_{tot}}{2 \cdot \omega_1 \cdot m_{tot}}$$

*

$$\frac{\xi_1^{SPD}}{\xi_1^{MPD}} \cong \frac{2}{N(N+1)}$$

$$\xi_1^{MPD} \cong \xi_1^{SPD} \frac{N(N+1)}{2}$$

Properties of MPD and SPD systems (9)

If a target damping ratio is looked for:
the **fundamental results** are:

$$\xi_1^{SPD} = \bar{\xi}$$

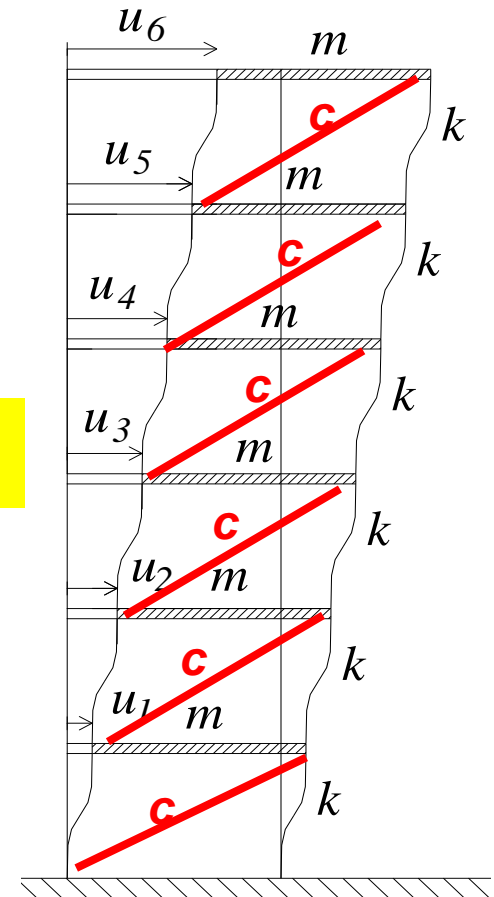
$$c_{tot,SPD} = \bar{\xi} \cdot \omega_1 \cdot m_{tot} \cdot N(N+1)$$

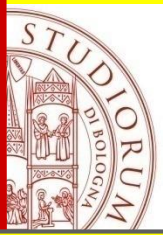
$$c_{storey,SPD} = \frac{c_{tot,SPD}}{N} = \bar{\xi} \cdot \omega_1 \cdot m_{tot} \cdot (N+1)$$

**

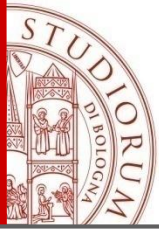
The above equations allow to size the damping coefficients of each damper of an inter-storey dampers system, in order to get a target damping ratio $\bar{\xi}$, by **simply knowing:**

- the total mass m_{tot}
- the fundamental period of vibration T_1 (or ω_1)
- the total number of storeys N





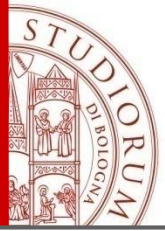
The five-step procedure (2010)



Design strategy

- The design philosophy is **to limit the structural damages** under severe earthquakes.
- The structural elements (columns and beams) should remain **in the elastic phase**.
- Let's keep the **ductility resources** of columns and beams **as an additional property** to withstand very severe and unexpected earthquakes.

The five-step procedure for inter-storey dampers placement



DESIGN PROCESS

VERIFICATION

STEP 1: performance objectives



$$\bar{\eta} \rightarrow \bar{\xi}$$

$$\bar{\eta} = \sqrt{\frac{10}{5 + \bar{\xi}}}$$

STEP 2: linear dampers



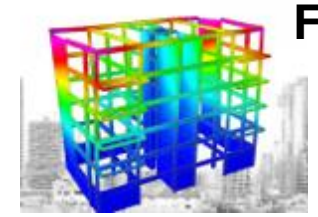
$$\bar{\xi} \rightarrow \bar{c}_L$$

$$\bar{c}_L = \bar{\xi} \cdot \omega_1 \cdot m_{tot} \cdot \left(\frac{N+1}{n} \right)$$

STEP 3: linear TH analyses



structural response



FEM

v_{max}

STEP 4: non-linear dampers



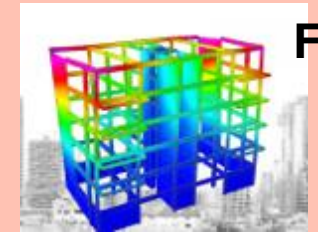
$$\bar{c}_L \rightarrow \begin{cases} \bar{c}_{NL} \\ \bar{\alpha} \\ \bar{k}_{oil} \end{cases}$$

$$\bar{c}_{NL} = \bar{c}_L \cdot (\chi \cdot v_{max})^{1-\alpha}$$

STEP 5: non-linear TH analyses



structural response



FEM



The five-step procedure for inter-storey dampers placement

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A Five-Step Procedure for the Dimensioning of Viscous Dampers to Be Inserted in Building Structures

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and TOMASO TROMBETTI

Department DISTART, University of Bologna, Bologna, Italy

Viscous dampers have widely proved their effectiveness in mitigating the effects of the seismic action upon building structures. In view of the large impact that use of such dissipative devices is already having and would most likely have soon in earthquake engineering applications, this article presents a practical procedure for the seismic design of building structures equipped with viscous dampers, which aims at providing practical tools for an easy identification of the mechanical characteristics of the manufactured viscous dampers which allow to achieve target levels of performances. Selected numerical applications are developed with reference to simple, but yet relevant, cases.

Keywords Added Viscous Dampers; Seismic Design; Design Procedure; Nonlinear Modeling; Damping Ratio

Starting point

 $\bar{\eta}$

$$\bar{\eta} = \frac{V_{base,\xi}}{V_{base,\xi=5\%}}$$

$$\bar{\eta} = \frac{M_{Ed,\xi}}{M_{Ed,\xi=5\%}}$$

$$\bar{\eta} = \frac{\delta_{top-storey,\xi}}{\delta_{top-storey,\xi=5\%}}$$

EXAMPLE:

If the bending moment action at the base of a column is $M_{Ed,\xi=5\%} = 1000$ kNm and if the bending moment resistance is $M_{Rd} = 500$ kNm

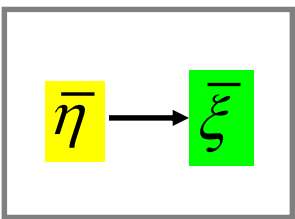
then

we want that dampers lead to $M_{Ed,\xi} = 500$ kNm

$$\bar{\eta} = \frac{500}{1000} = 0.5$$

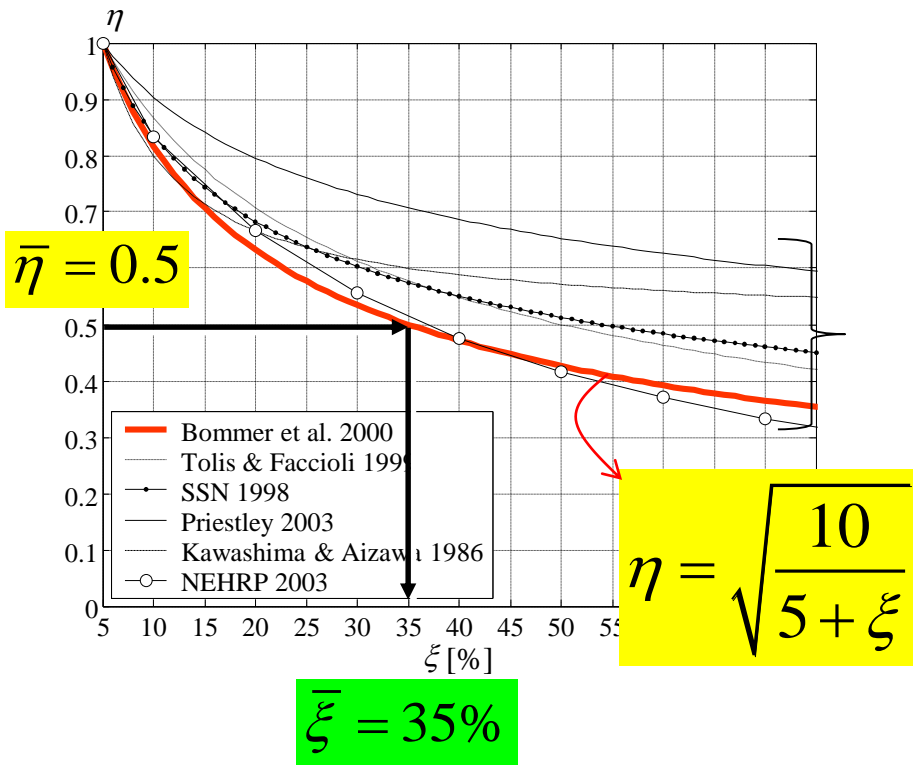
$$\bar{\eta} = \frac{\text{target seismic demand (i.e. capacity/strength or acceptable action or drift)}}{\text{actual seismic demand with no dampers}}$$

Step 1

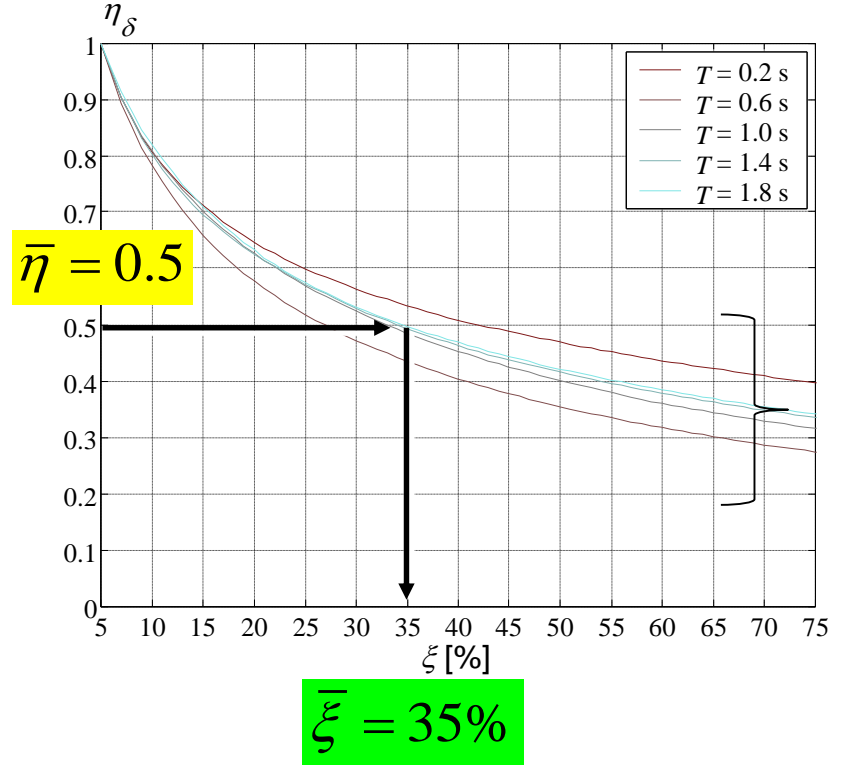


Some available formulations to relate $\bar{\eta} \rightarrow \bar{\xi}$

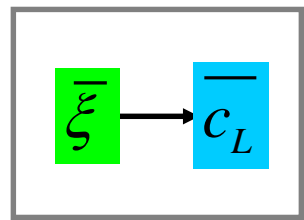
Cardone, Dolce, Rivelli (ANIDIS 2007)



mean SDOF response from T-H analyses

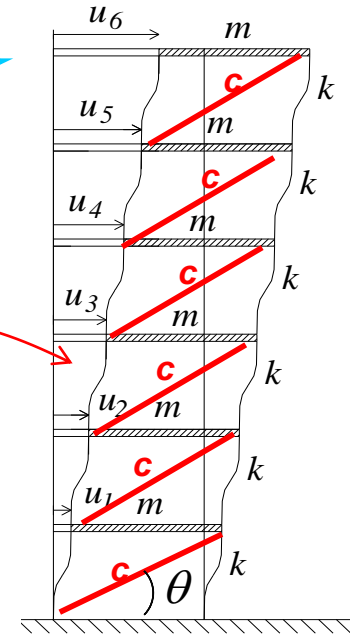
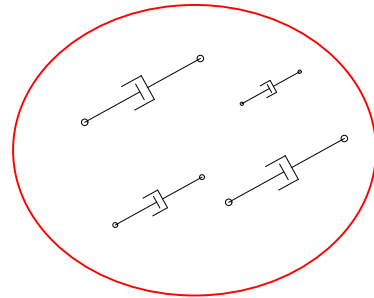


Step 2



Preliminary sizing of linear viscous damper using analytical formula **

$$\left. \begin{array}{l} c_L \\ \alpha = 1 \\ k_{oil} = \infty \end{array} \right\}$$



$$c_L = \bar{\xi} \cdot \omega_1 \cdot m_{tot} \cdot \frac{(N+1)}{n}$$

**

$$c_{L,inclined} = c_{L,horizontal} \cdot \left(\frac{1}{\cos^2 \theta} \right)$$

n = number of dampers at a given storey in a given direction

Step 3

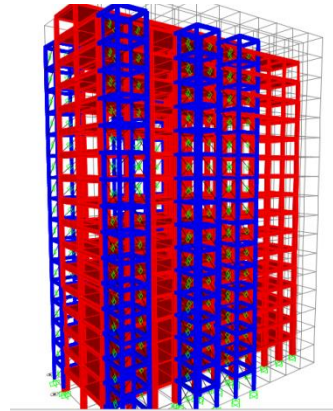
FEM
simulations

After linear viscous dampers are dimensioned

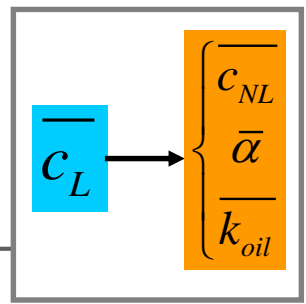
$$c_L = \overline{c_L} \quad , \quad \alpha = 1 \quad , \quad k_{oil} = \infty$$

Linear TH dynamic analyses are necessary in order to:

1. Verify by means of **snap-back tests** that the actual damping properties of the model are in line with the expected ones (e.g. **target damping ratio is achieved**) $\overline{\xi}$
2. Calculate the **maximum working velocities of the linear dampers** v_{max}
3. Calculate **the maximum damper piston-strokes** x_{max}



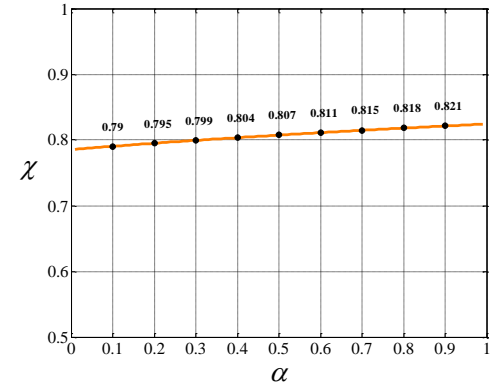
Step 4



ENERGETIC APPROACH:

equal energy dissipated over a full cycle of harmonic motion

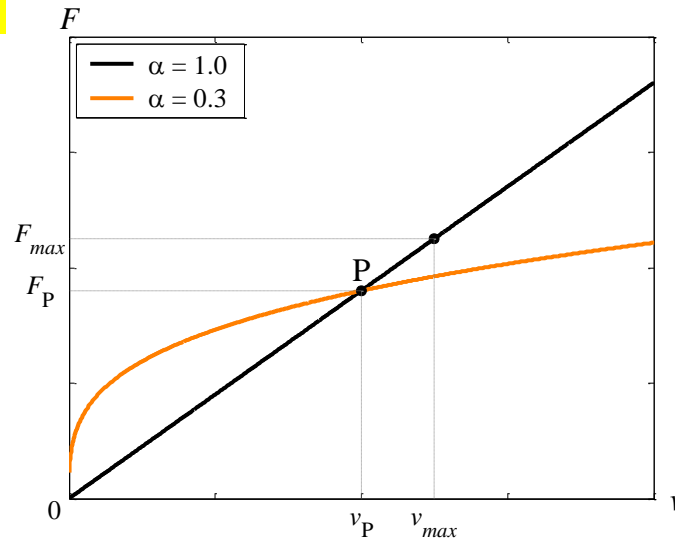
$$\overline{E_{d,L}} = \overline{E_{d,NL}} \quad \longrightarrow \quad \overline{c_{NL}} = \overline{c_L} \cdot \left(\chi \cdot v_{\max} \right)^{1-\overline{\alpha}}$$



REFERENCE POINT P:

$$v_P = 0.8 \cdot v_{\max}$$

$$F_P = 0.8 \cdot \overline{c_L} \cdot v_{\max}$$



$$\chi = \left(\frac{\sqrt{\pi}}{2} \frac{\Gamma\left(\frac{\overline{\alpha}+3}{2}\right)}{\Gamma\left(\frac{\overline{\alpha}+2}{2}\right)} \right)^{\frac{1}{1-\overline{\alpha}}} \cong 0.8, \quad \forall \alpha$$

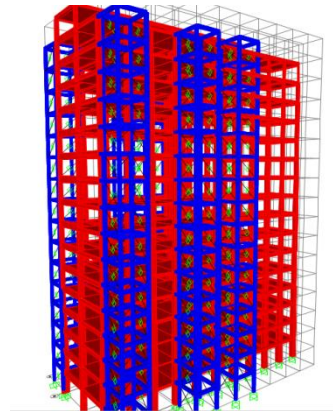
Step 5

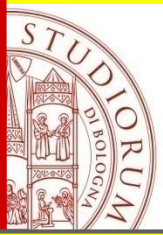
After non-linear viscous dampers are dimensioned

$$c_{NL} = \overline{c_{NL}} \quad , \quad \alpha = \overline{\alpha} \quad , \quad k_{oil} = \overline{k_{oil}} \geq 10 \cdot c_L \cdot \omega_1$$

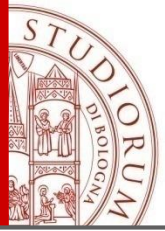
Non-linear TH dynamic analyses are necessary in order to:

1. Verify the effectiveness of the non-linear dampers in reducing the global structural response (e.g. **target damping ratio is achieved, maximum forces in the structural elements are acceptable**)
2. Evaluate the **maximum damper forces in the non-linear dampers**
3. Evaluate **the maximum strokes in the non-linear dampers**



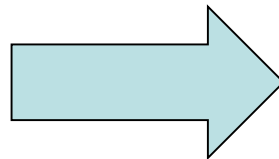
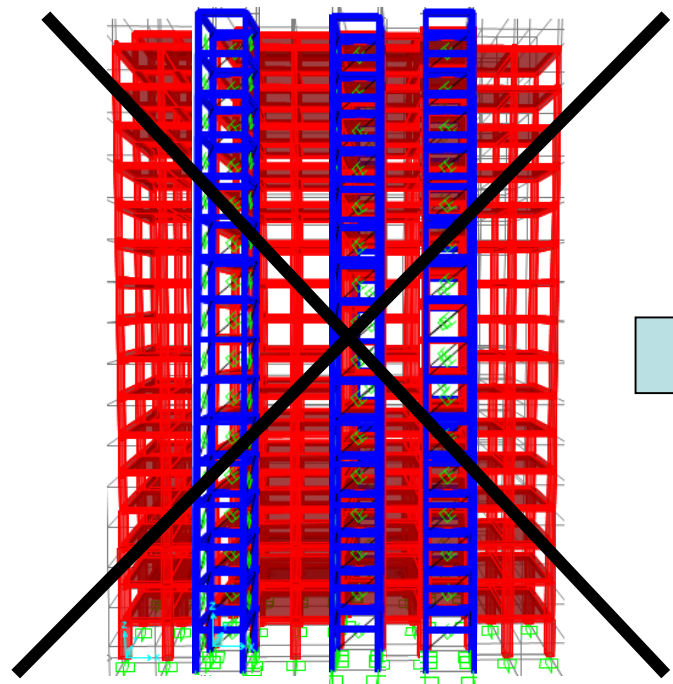


The direct five-step procedure (2016-2018)

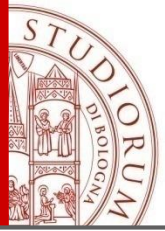


A step forward: a direct five-step procedure

The challenge:
Can we directly design (at least PRELIMINAR SIZING) the viscous dampers and the frames without performing TH dynamic analyses?



The five-step procedure for inter-storey dampers placement



STEP 1: performance objectives



$$\bar{\eta} \rightarrow \bar{\xi}$$

$$\bar{\eta} = \sqrt{\frac{10}{5 + \bar{\xi}}}$$

STEP 2: linear dampers



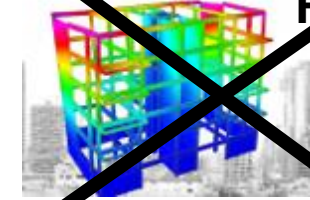
$$\bar{\xi} \rightarrow \bar{c}_L$$

$$\bar{c}_L = \bar{\xi} \cdot \omega_1 \cdot m_{tot} \cdot \left(\frac{N+1}{n} \right)$$

STEP 3: linear TH analyses



structural response



FEM

v_{max}

STEP 4: non-linear dampers



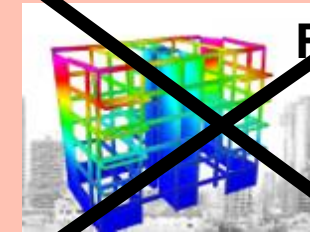
$$\bar{c}_L \rightarrow \begin{cases} \bar{c}_{NL} \\ \bar{\alpha} \\ \bar{k}_{oil} \end{cases}$$

$$\bar{c}_{NL} = \bar{c}_L \cdot (\chi \cdot v_{max})^{1-\alpha}$$

STEP 5: non-linear TH analyses



structural response

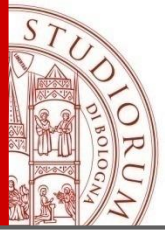


FEM

DESIGN PROCESS

VERIFICATION

The five-step procedure for inter-storey dampers placement



DESIGN PROCESS

VERIFICATION

STEP 1: performance objectives



$$\bar{\eta} \rightarrow \bar{\xi}$$

$$\bar{\eta} = \sqrt{\frac{10}{5 + \bar{\xi}}}$$

STEP 2: linear dampers




$$\bar{\xi} \rightarrow \bar{c}_L$$

$$c_L = \bar{\xi} \cdot \omega_1 \cdot m_{tot} \cdot \left(\frac{N+1}{n} \right)$$

STEP 3: linear TH analyses



structural response



1. Estimation of inter-storey velocities

STEP 4: non-linear dampers




$$\bar{c}_L \rightarrow \begin{cases} c_{NL} \\ \bar{\alpha} \\ k_{oil} \end{cases}$$

$$c_{NL} = c_L \cdot (\chi \cdot v_{max})^{1-\alpha}$$

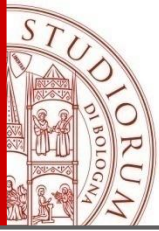
STEP 5: non-linear TH analyses



structural response

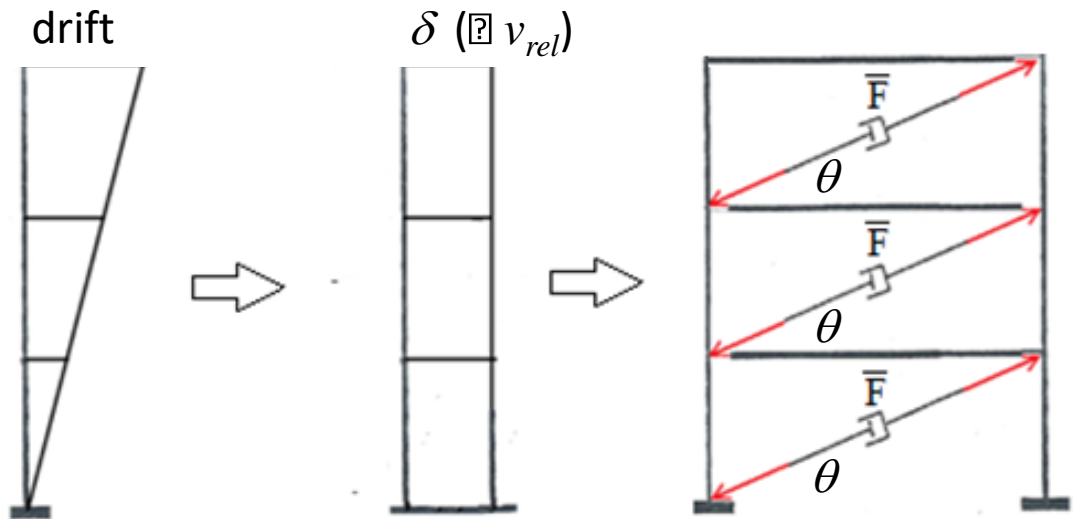
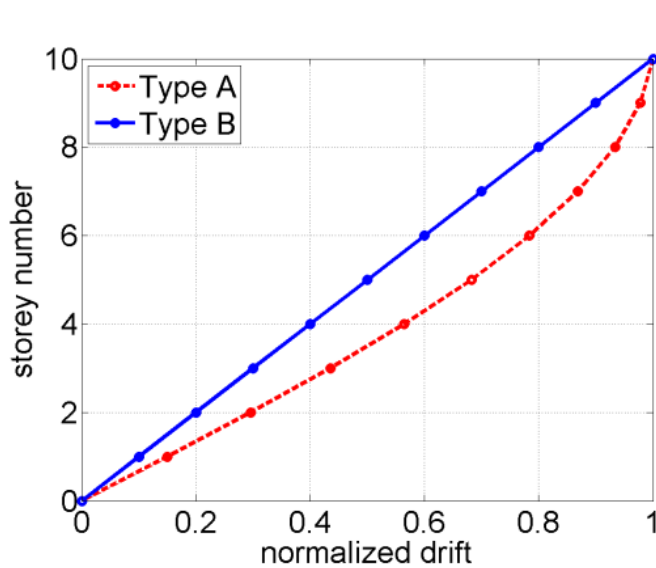


2. Equivalent Static Analysis (ESA)



1. Inter-storey velocities and damper forces

Analytical Estimations Based On First Mode Response



$$\mathbf{B} \quad \delta_{\max} = \frac{S_{e,\xi}(T_1)}{\omega_1^2} \cdot \frac{2}{(N+1)}$$

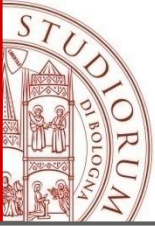
$$v_{\max} = \frac{S_{e,\xi}(T_1)}{\omega_1} \cdot \frac{2}{(N+1)} \cdot \cos \theta$$

$$F_{D\max} = \frac{2 \cdot \xi \cdot m_{tot} \cdot S_{e,\xi}(T_1)}{\cos \theta}$$

$$\mathbf{A} \quad \delta_{\max} = \frac{S_{e,\xi}(T_1)}{\omega_1^2} \cdot \frac{12N}{(2 + 5N + 5N^2)}$$

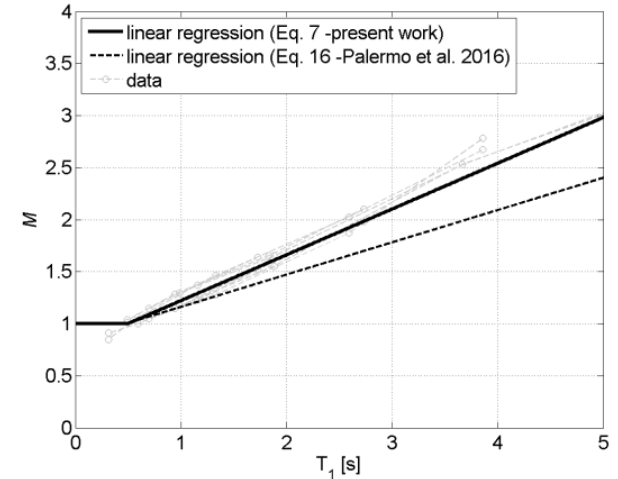
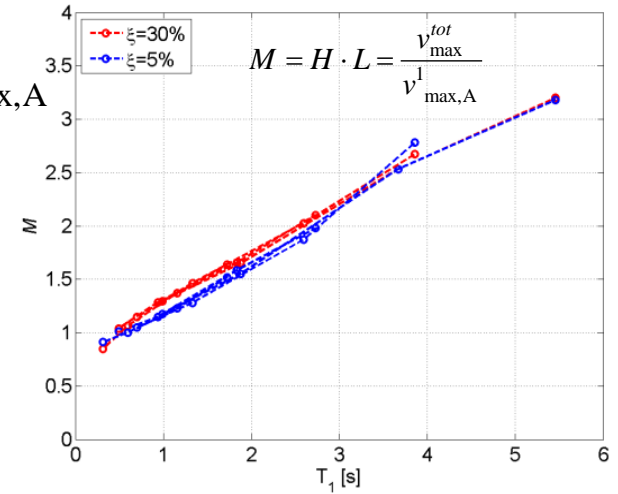
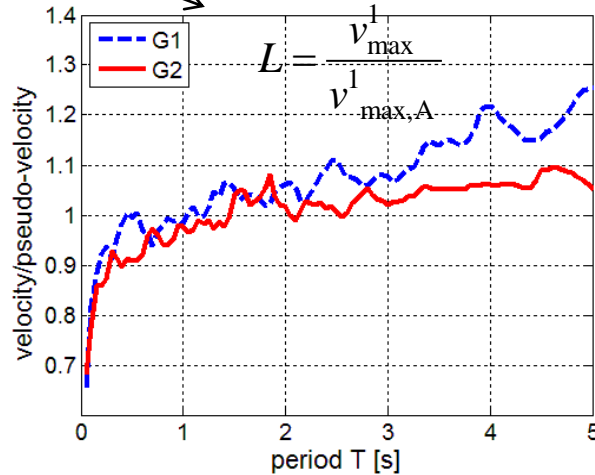
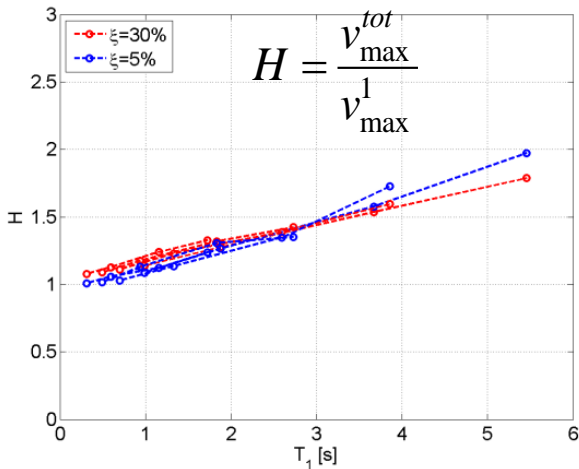
...

Palermo et al. 2016 (BEE)



1. Higher modes contribution

$$v_{\max}^{tot} = \left(\frac{v_{\max}^{tot}}{v_{\max}^1} \right) \cdot \left(\frac{v_{\max}^1}{v_{\max,A}^1} \right) \cdot v_{\max,A}^1 = H \cdot L \cdot v_{\max,A}^1 = M \cdot v_{\max,A}^1$$



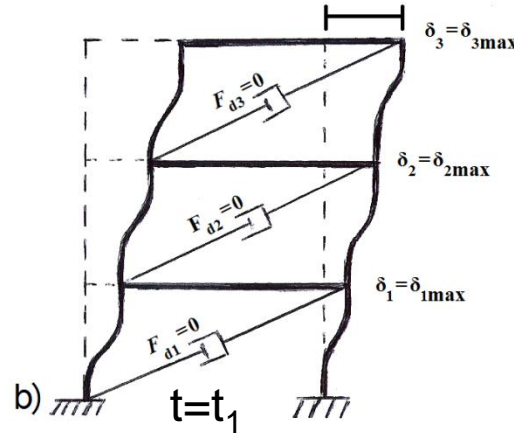
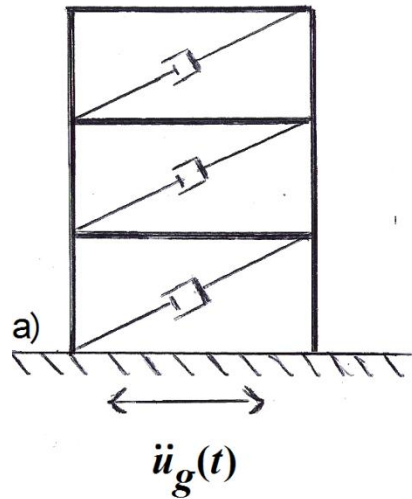
$$v_{\max,A}^{tot} = \begin{cases} \frac{12N}{(2+5N+5N^2)} \cdot \frac{S_a(T_1)}{\omega_1} & \text{for } T_1 \leq 0.5s \\ (0.44 \cdot T_1 + 0.78) \cdot \frac{12N}{(2+5N+5N^2)} \cdot \frac{S_a(T_1)}{\omega_1} & \text{for } 0.5 < T_1 \leq 5.0s \end{cases}$$

correction factor M

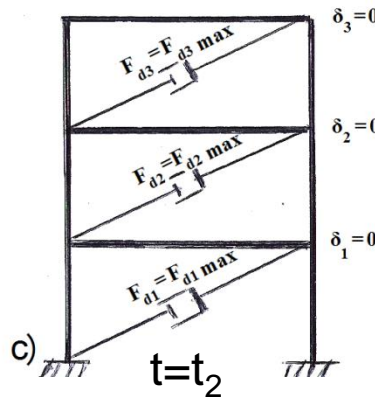
Palermo et al. 2017 (SDEE)



2. Equivalent Static Analysis for damped structures

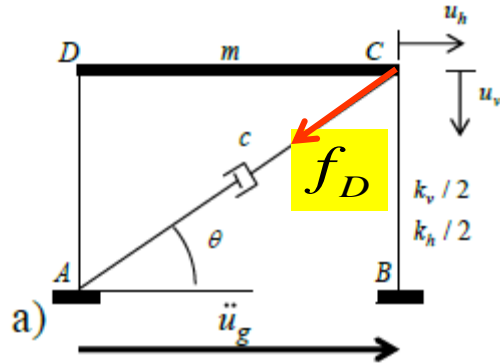


Maximum Lateral Displacement Configuration (ESA1)



Maximum Velocity (Max Damper Force) Configuration (ESA2)

2. The rationale behind ESA: the damped frame



dynamic equations

$$\begin{cases} f_{Ih} + f_{Dh} + f_{Sh} = 0 \\ f_{Iv} + f_{Dv} + f_{Sv} = 0 \end{cases}$$

two coupled equations (due to the damper)

$$\begin{cases} m\ddot{u}_h + c_h\dot{u}_h + c_v\dot{u}_v + k_h u_h = -m\ddot{u}_g(t) \\ m\ddot{u}_v + c_{hv}\dot{u}_h + c_v\dot{u}_v + k_v u_v = 0 \end{cases}$$

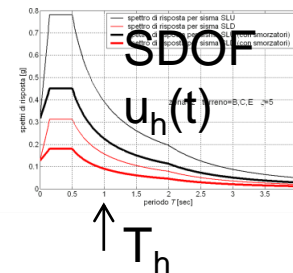


$$k_v \ll k_h$$

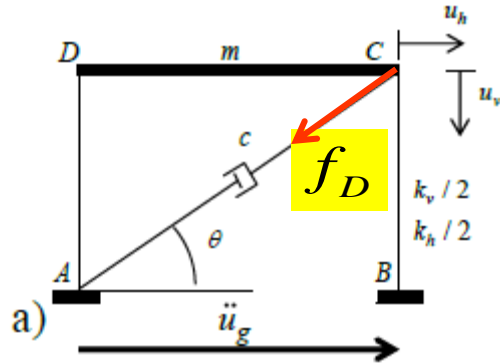
\dot{u}_v can be neglected in the first equation

first equation
(only in the horizontal d.o.f.)

$$m\ddot{u}_h + c_h\dot{u}_h + k_h u_h = -m\ddot{u}_g(t)$$



2. The rationale behind ESA: the damped frame



u_h is obtained
and the damper force is obtained as:

$$c_h \dot{u}_h(t) = f_{Dh}(t) = f_D(t) \cdot \cos \theta$$

$$f_D(t) = \frac{c_h \dot{u}_h(t)}{\cos \theta}$$

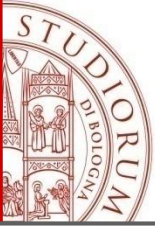


second
equation

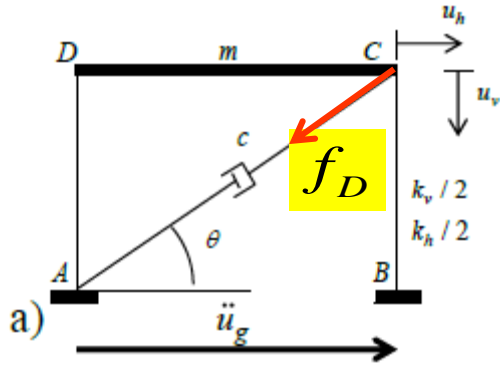
$$m \ddot{u}_v + c_v \dot{u}_v + k_v u_v = -c_{hv} \dot{u}_h = -f_{Dv}(t) = -f_D(t) \cdot \sin \theta$$

$$m \ddot{u}_v + c_v \dot{u}_v + k_v u_v = -f_{Dv}(t)$$

the input for the vertical
d.o.f. is given by the
damper force (coupled
response)

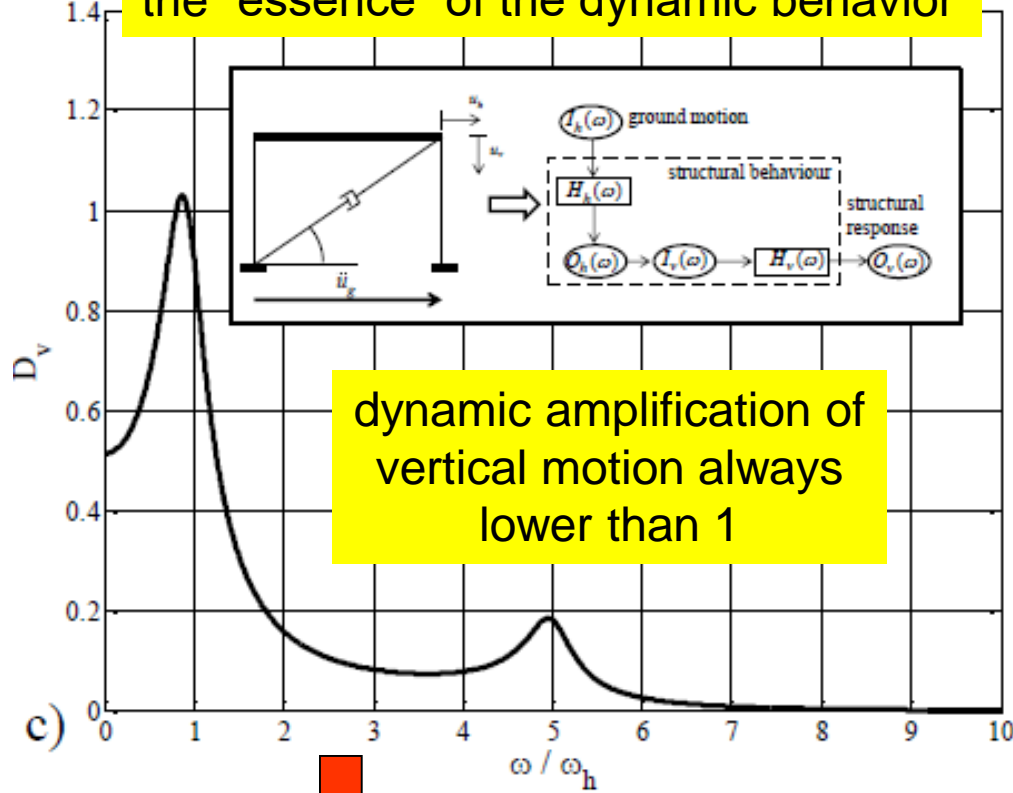


2. The rationale behind ESA: the damped frame



$$k_v \ll k_h$$

the "essence" of the dynamic behavior



dynamic amplification of vertical motion always lower than 1

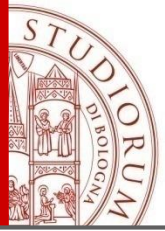
the second dynamic equation can be treated as a static equation

$$m \cancel{u_v} + c \cancel{\dot{u}_v} + k_v u_v = -f_{Dv}(t)$$

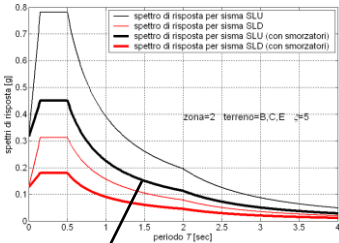
axial force in the column

$$k_v u_{v,max} = -f_{Dv,max}$$

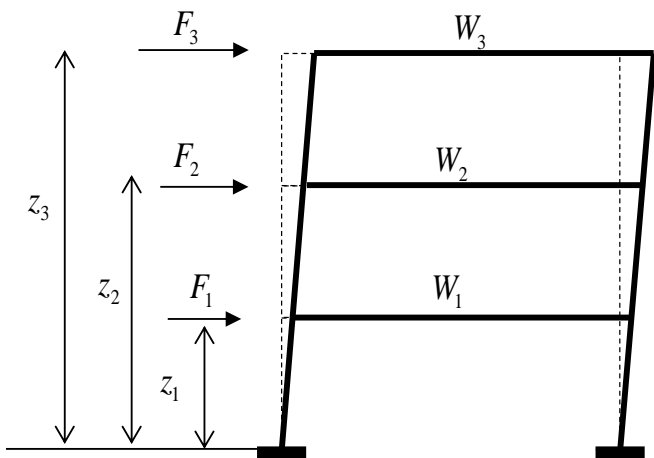
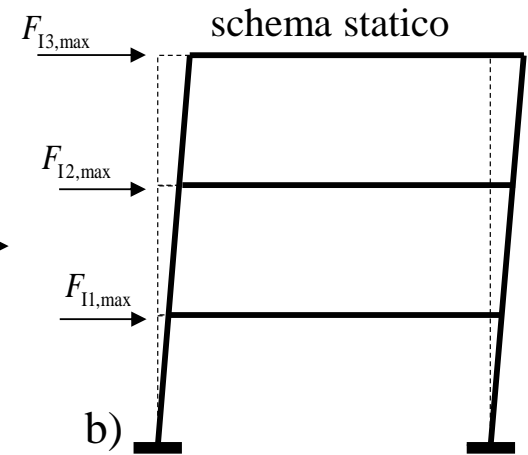
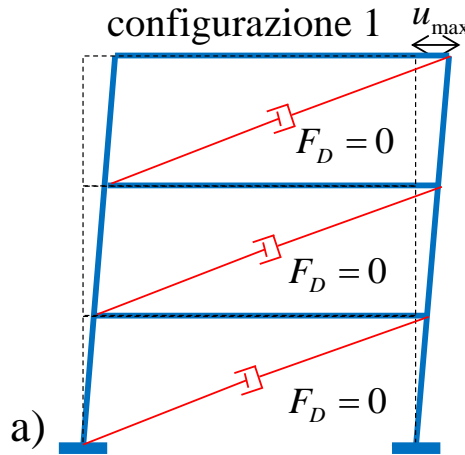
Palermo et al. 2018 (BEE)



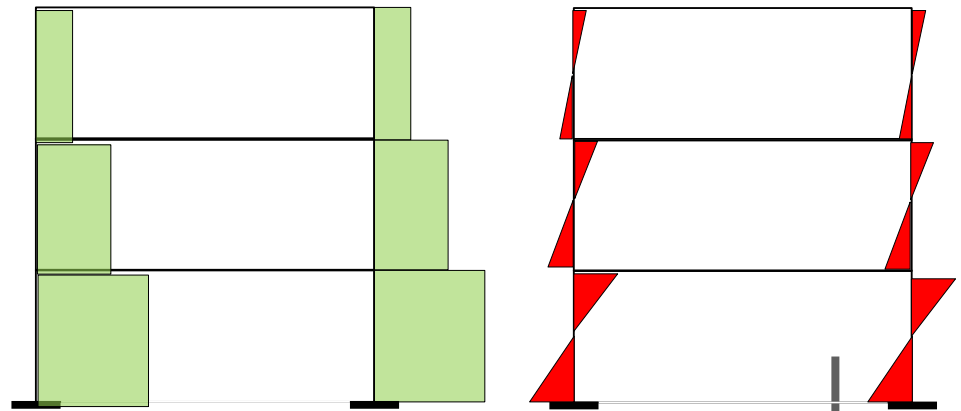
How to perform Equivalent Static Analysis ESA1

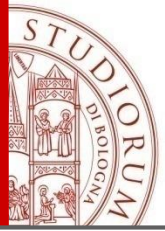


$$F_i = S_{e,\xi} \cdot \frac{W_i}{\sum_{i=1,2,\dots,N} W_i \cdot z_i}$$

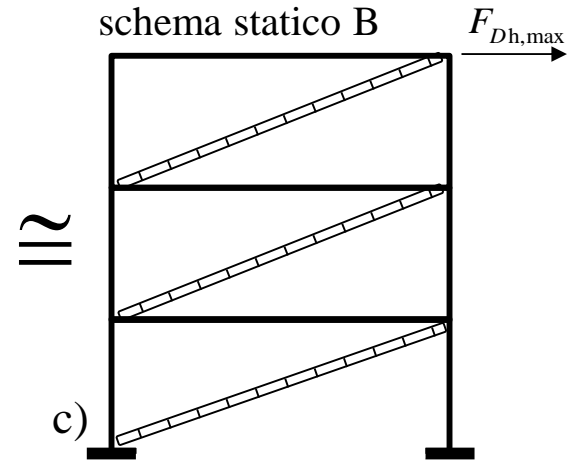
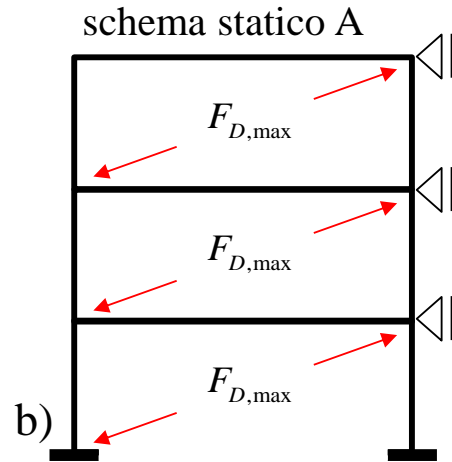
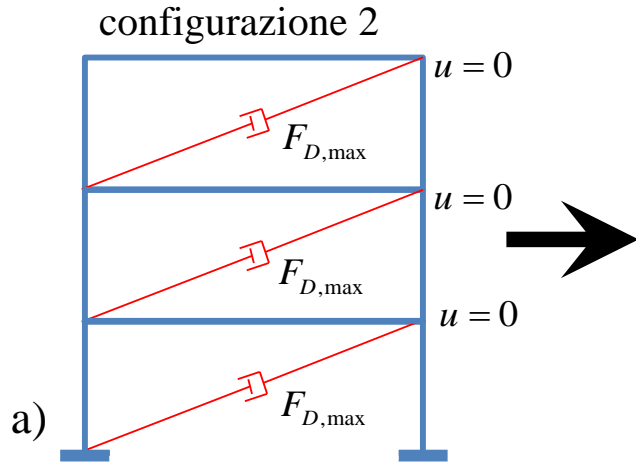


taglio
 momento flettente

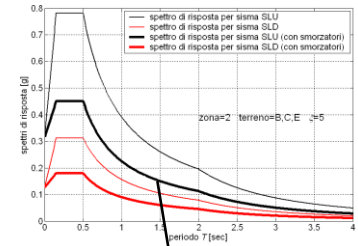
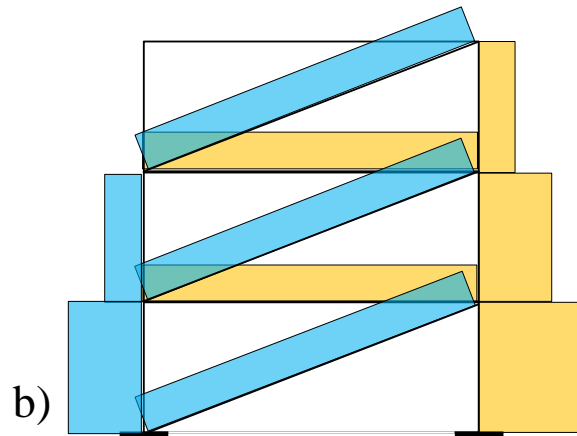
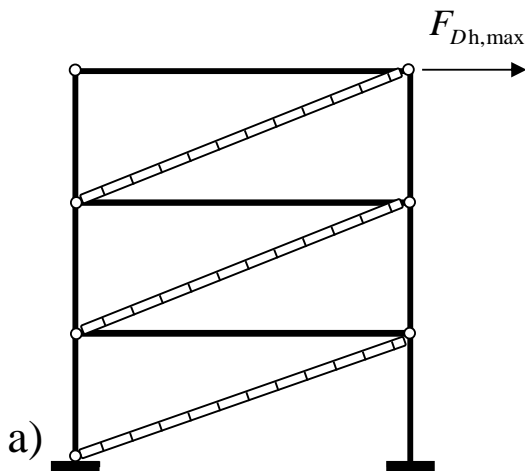




How to perform Equivalent Static Analysis ESA2



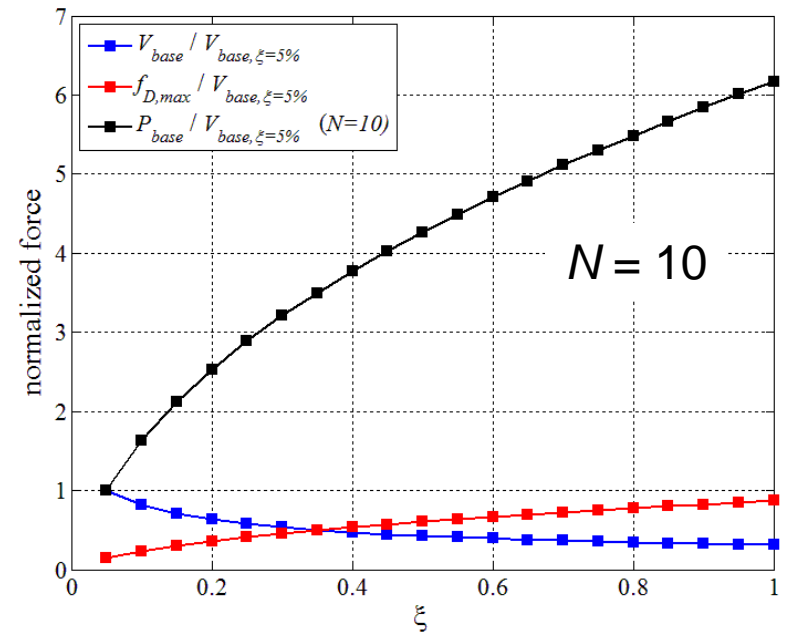
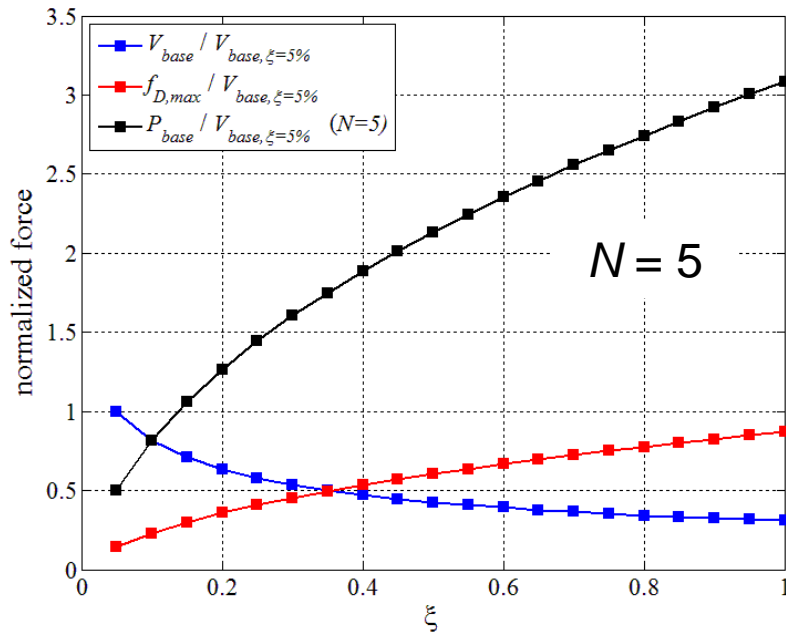
sforzo assiale (compressione)
 sforzo assiale (trazione)



COLUMNS AXIAL FORCES DUE TO DAMPERS

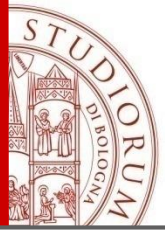
$$\begin{aligned}
 P_{i,max} &= (N - i + 1) F_{d,v,max} \\
 &= (N - i + 1) \cdot \frac{2 \cdot \xi \cdot W_{tot}}{n} S_{e,\xi} \tan \theta
 \end{aligned}$$

Axial forces in the columns due to dampers



REMEMBER: For tall buildings the axial forces due to dampers may become even larger than the axial forces due to static vertical loads

The direct five-step procedure for inter-storey dampers placement



DESIGN PROCESS

STEP 1: performance objectives



$$\bar{\eta} \rightarrow \bar{\xi}$$

$$\bar{\eta} = \sqrt{\frac{10}{5 + \bar{\xi}}}$$

STEP 2: linear damper

$$v_{\max} = \frac{S_{e,\xi}(T_1)}{\omega_1} \cdot \frac{2}{(N+1)} \cdot \cos \theta$$

$$c_L = \bar{\xi} \cdot \omega_1 \cdot m_{tot} \cdot \left(\frac{N+1}{n} \right)$$

STEP 3: linear TH

analytical estimations

response

$$v_{\max} = \frac{S_{e,\xi}(T_1)}{\omega_1} \cdot \frac{2}{(N+1)} \cdot \cos \theta$$

$$F_{D\max} = \frac{2 \cdot \xi \cdot m_{tot} \cdot S_{e,\xi}(T_1)}{\cos \theta}$$

STEP 4: non-linear

$$F_{D\max} = \frac{2 \cdot \xi \cdot m_{tot} \cdot S_{e,\xi}(T_1)}{\cos \theta}$$

$$c_{NL} = c_L \cdot (\chi \cdot v_{\max})^{1-\alpha}$$

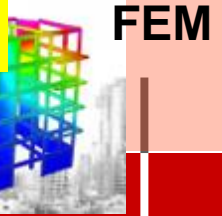
(k_{oil})

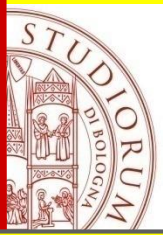
VERIFICATION

STEP 5:

$$P_{i,\max} = (N - i + 1) \cdot \frac{2 \cdot \xi \cdot W_{tot} \cdot S_{e,\xi}}{n} \cdot \tan \theta$$

$$\frac{2 \cdot \xi \cdot W_{tot} \cdot S_{e,\xi}}{n} \cdot \tan \theta$$





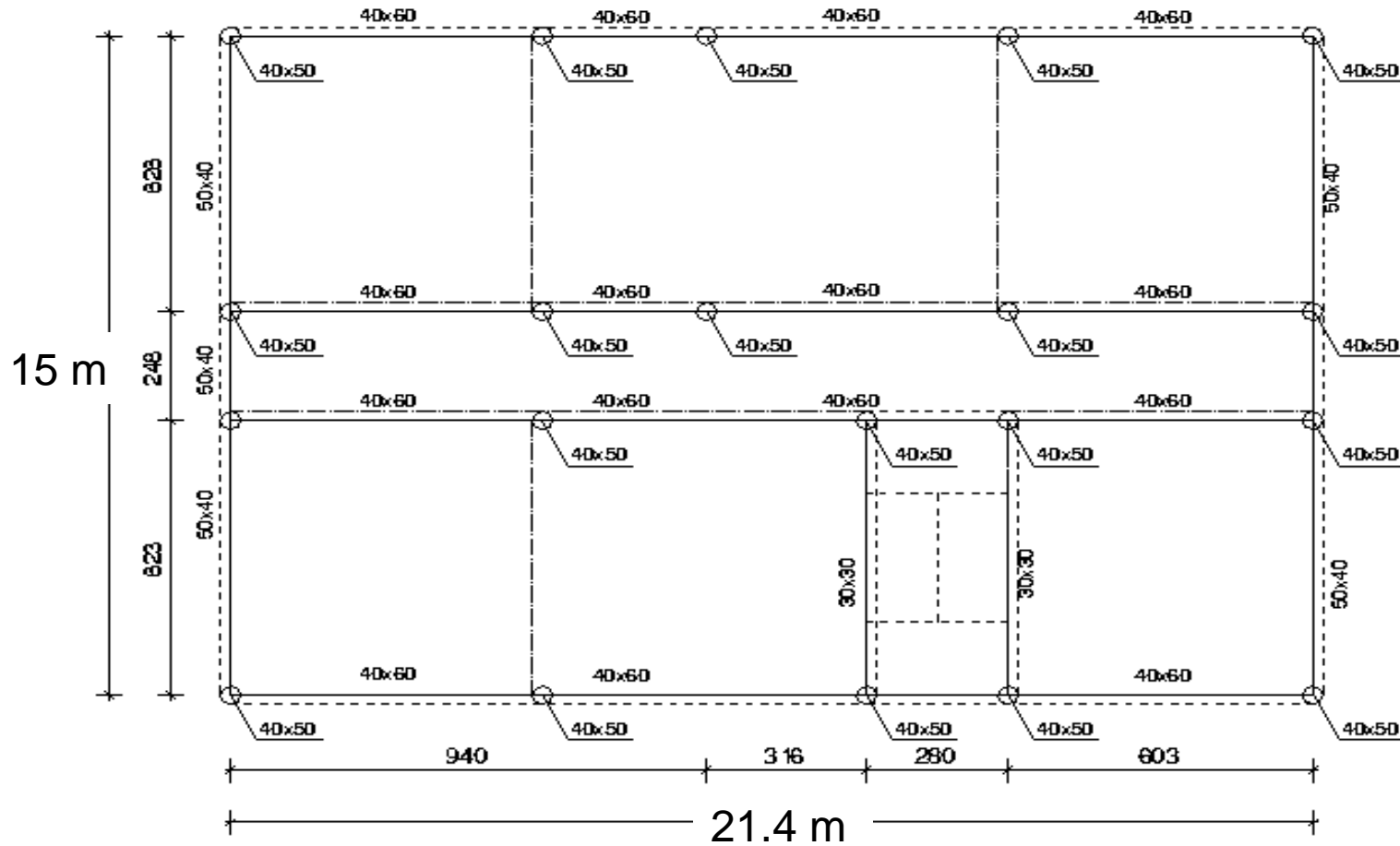
Applicative example

The case study

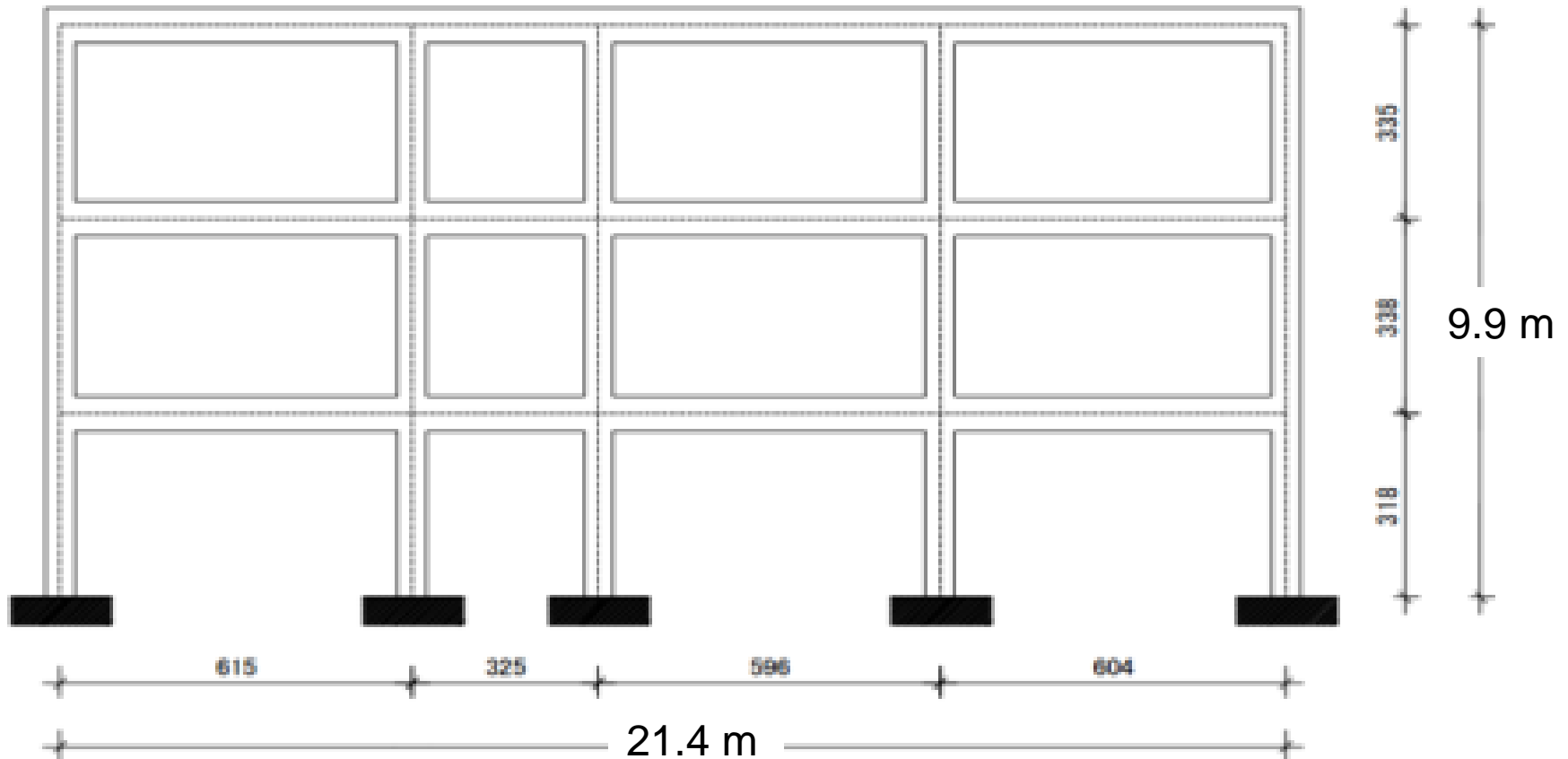


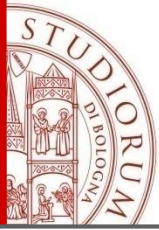
reinforced-concrete 3-storey school building
located in Bisignano (Cosenza, southern Italy)

The case study



The case study





The case study

Table 1: Load analysis

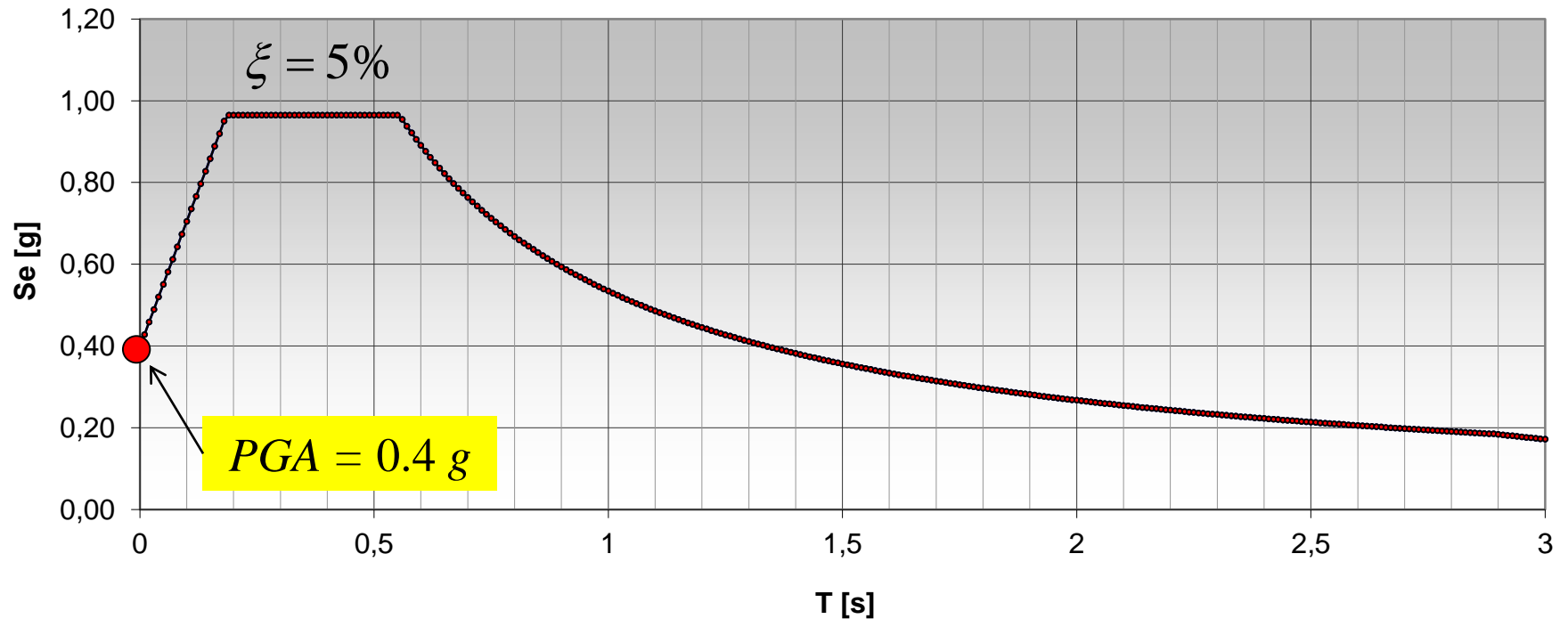
<i>Loads</i>	<i>Floor 1</i>	<i>Floor 2</i>	<i>Floor 3 (attic + roof)</i>
Permanent G_1	3.00 kN/m ²	3.00 kN/m ²	4.00 kN/m ²
Permanent G_2	2.00 kN/m ²	2.00 kN/m ²	3.00 kN/m ²
Imposed Loads Q	3.00 kN/m ² ($\Psi_2=0.6$)	3.00 kN/m ² ($\Psi_2=0.6$)	2.50 kN/m ² ($\Psi_2=0$)
TOTAL in static conditions	8.00 kN/m ²	8.00 kN/m ²	9.50 kN/m ²
TOTAL in seismic conditions	6.80 kN/m ²	6.80 kN/m ²	7.00 kN/m ²

total weight in seismic conditions

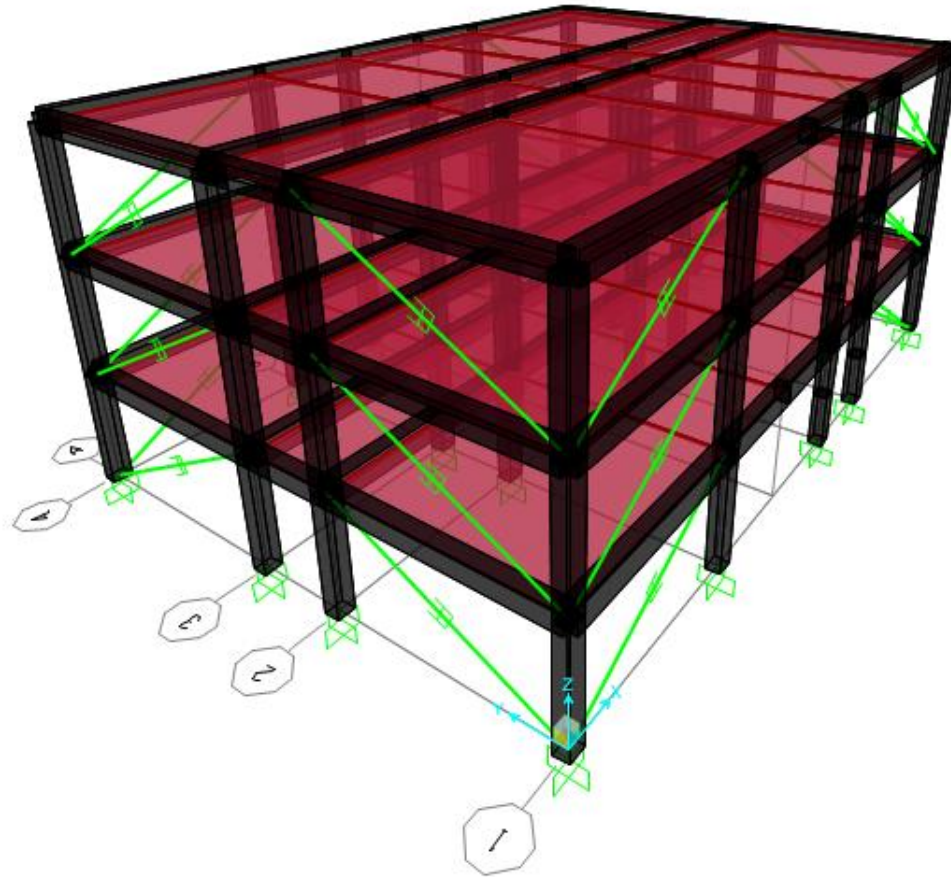
$$W_{tot} = 11900 \text{ kN}$$

The case study

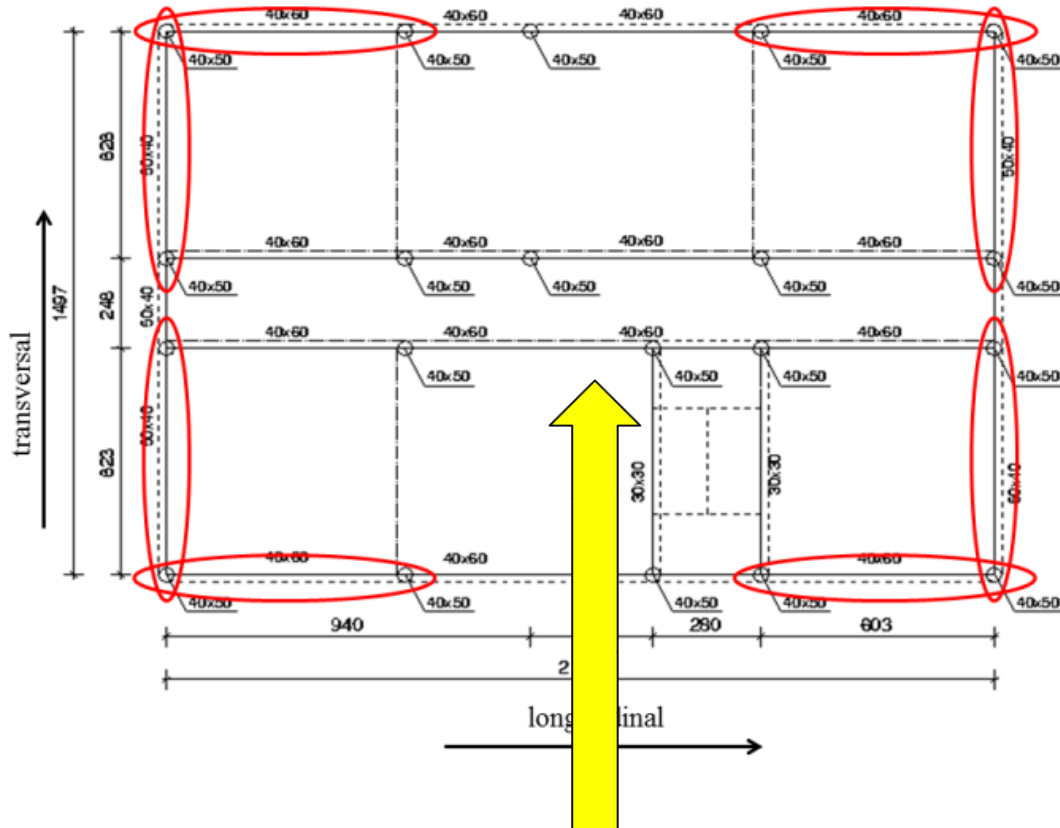
the horizontal pseudo-acceleration elastic response spectrum



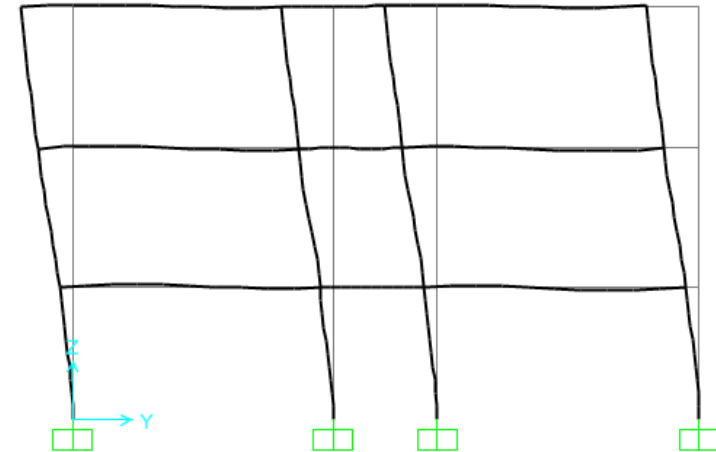
The case study



The direct five-step procedure applied to the case study

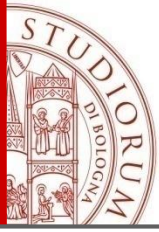


seismic action
along the transversal direction



first mode
along the transversal direction

$$T_{1,transversal} = 0.80 \text{ s}$$



Application - Step 1

STEP 1

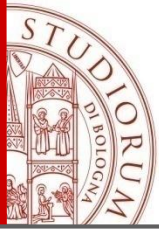
Assumed target damping ratio: $\bar{\xi}_{tot} = \xi_{intr} + \bar{\xi}_{visc} = 5\% + 20\% = 25\%$

through dampers

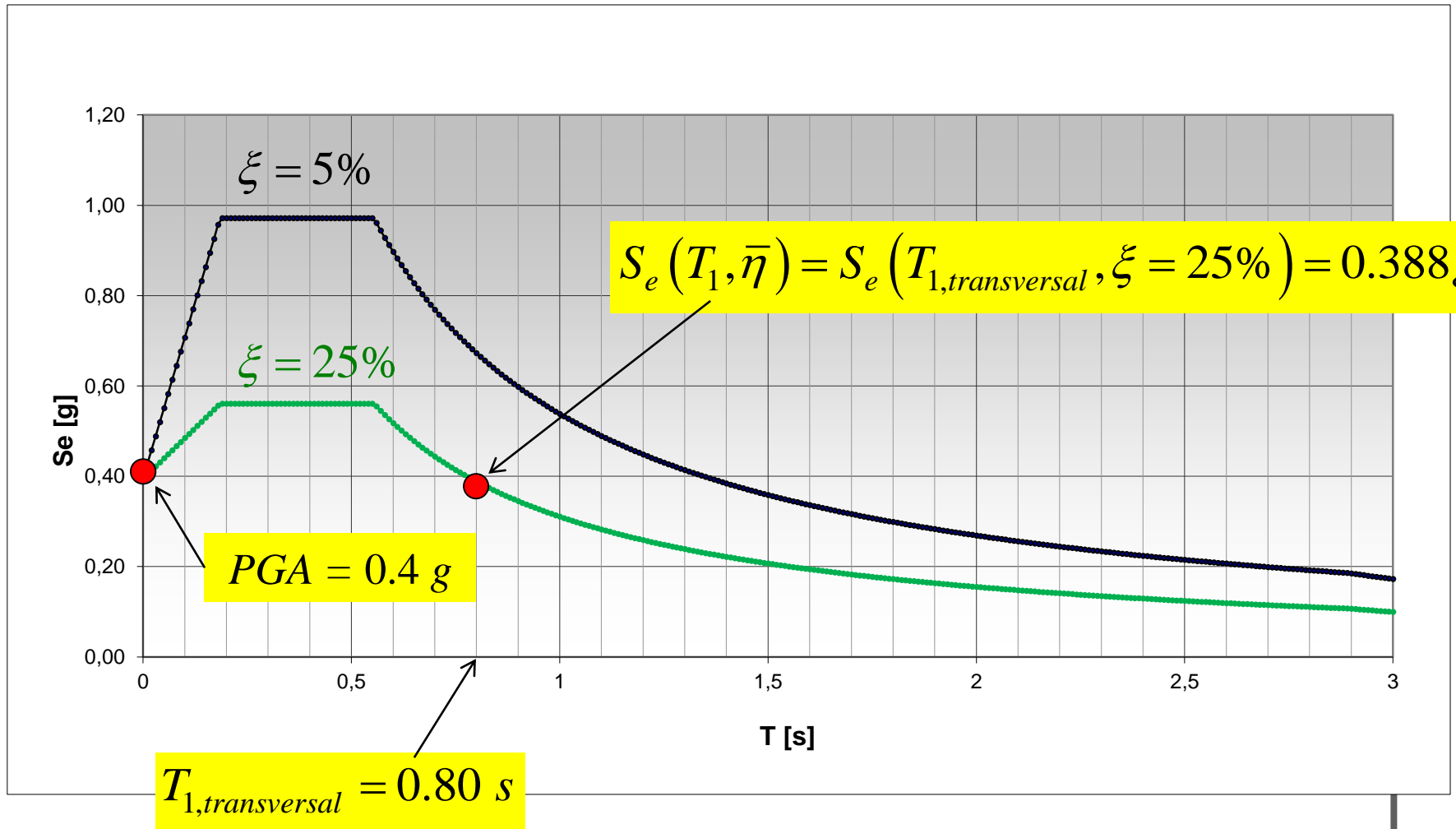
Corresponding response reduction factor: $\bar{\eta} = \sqrt{\frac{10}{5 + \bar{\xi}_{tot}}} = \sqrt{\frac{10}{5 + 25}} = 0.577$

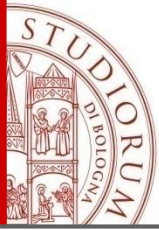
Fundamental period along the considered (transversal) direction: $T_1 = 0.80s$

Spectral acceleration: $S_e(T_1, \bar{\eta}) = a_g \cdot S \cdot \bar{\eta} \cdot F_o \cdot \left(\frac{T_c}{T_1}\right) =$
 $= 0.323g \cdot 1.23 \cdot 0.577 \cdot 2.43 \cdot \frac{0.55}{0.80} = 0.388g$



Application – Step 1





Application - Step 2

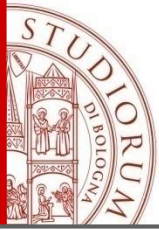
STEP 2

Number of dampers per floor placed along the longitudinal direction: $n = 4$

Damper inclination with respect to the horizontal line: $\theta = 28^\circ$

Linear damping coefficient:

$$\begin{aligned} c_L &= \bar{\zeta}_{visc} \cdot \omega_1 \cdot \frac{W_{tot}}{g} \cdot \left(\frac{N+1}{n} \right) \frac{1}{\cos^2 \theta} = \\ &= 0.20 \cdot \frac{2\pi}{0.80s} \cdot \frac{11900 \text{ kN}}{9.81 \frac{\text{m}}{\text{s}^2}} \cdot \left(\frac{3+1}{4} \right) \cdot \frac{1}{\cos^2 28^\circ} \cong 2444 \frac{\text{kN} \cdot \text{s}}{\text{m}} \end{aligned}$$

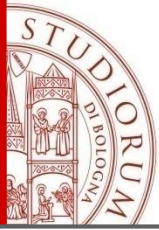


Application - Step 3

STEP 3

Peak damper **velocity** estimation for the equivalent linear damper:

$$\begin{aligned} v_{\max} &= \frac{S_e(T_1, \bar{\eta})}{\omega_1} \cdot \frac{2}{N+1} \cdot \cos \theta = \\ &= \frac{0.388 \cdot 9.81 \frac{\text{m}}{\text{s}^2}}{\left(\frac{2\pi}{0.80\text{s}} \right)} \cdot \frac{2}{3+1} \cdot \cos 28^\circ \cong 0.214 \frac{\text{m}}{\text{s}} \end{aligned}$$

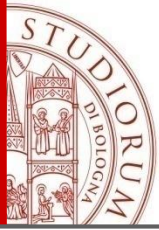


Application - Step 3

STEP 3

Peak damper **force** estimation for the equivalent linear damper:

$$\begin{aligned} F_{L,\max} &= 2 \cdot \bar{\xi}_{visc} \cdot \frac{W}{g} \cdot \frac{S_e(T_1, \bar{\eta})}{n \cdot \cos \theta} = \\ &= 2 \cdot 0.20 \cdot \frac{11900 \text{ kN}}{g} \cdot \frac{0.388g}{4 \cdot \cos 28^\circ} \cong 524 \text{ kN} \end{aligned}$$

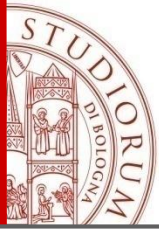


Application - Step 3

STEP 3

Peak damper **stroke** estimation for the equivalent linear damper:

$$\begin{aligned} s_{\max} &= \frac{S_e(T_1, \bar{\eta})}{\omega_1^2} \cdot \frac{2}{N+1} \cdot \cos \theta = \\ &= \frac{0.388g}{\left(\frac{2\pi}{0.80s}\right)^2} \cdot \frac{2}{3+1} \cdot \cos 28^\circ \cong 2.73\text{cm} \end{aligned}$$



Application - Step 4

STEP 4

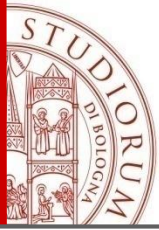
α -exponent of the commercial damper: $\alpha = 0.15$

Non-linear damping coefficient of the commercial damper:

$$\begin{aligned} c_{NL} &= c_L \cdot (0.8 \cdot v_{\max})^{1-\alpha} = \\ &= 2444 \frac{\text{kN} \cdot \text{s}}{\text{m}} \cdot \left(0.8 \cdot 0.214 \frac{\text{m}}{\text{s}} \right)^{1-0.15} \cong 546 \frac{\text{kN} \cdot \text{s}^{0.15}}{\text{m}^{0.15}} \end{aligned}$$

Minimum axial stiffness of the device (non-linear damper + supporting brace):

$$\begin{aligned} k_{axial} &\geq 10 \cdot c_L \cdot \omega_1 = \\ &= 10 \cdot 2444 \frac{\text{kN} \cdot \text{s}}{\text{m}} \cdot \frac{2\pi}{0.80\text{s}} = 1.92 \cdot 10^5 \frac{\text{kN}}{\text{m}} \longrightarrow 2 \cdot 10^5 \frac{\text{kN}}{\text{m}} \end{aligned}$$



Application - Step 4

STEP 4

Peak damper **force** estimation for the “non-linear” damper:

$$\begin{aligned} F_{NL,\max} &= 0.8^{1-\alpha} \cdot F_{L,\max} = \\ &= 0.8^{1-0.15} \cdot 524 \text{ kN} \cong 433 \text{ kN} \end{aligned}$$

Application - Step 5

$F_h \text{ tot} = 4623 \text{ kN}$

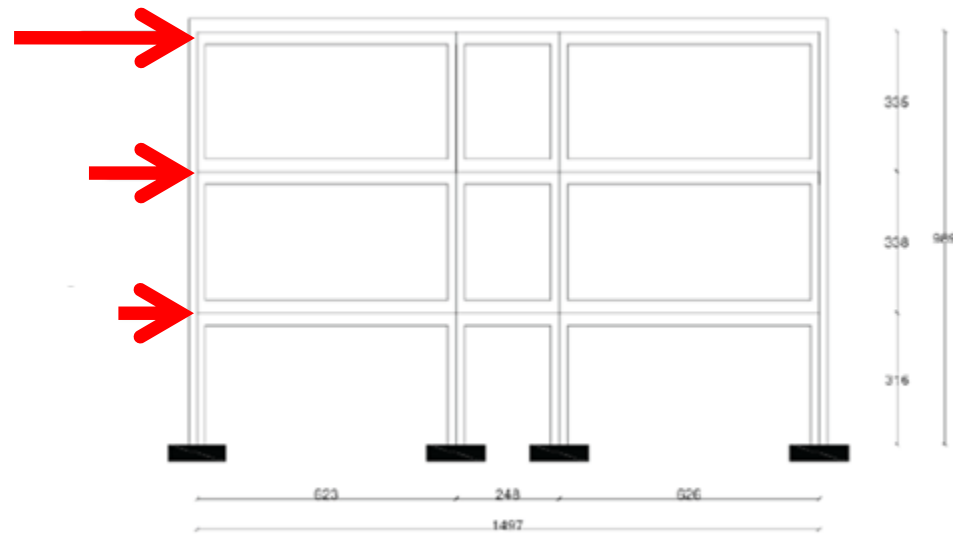


Figure 9. Static scheme to be solved for ESA1.

ESA1

382 kN

382 kN

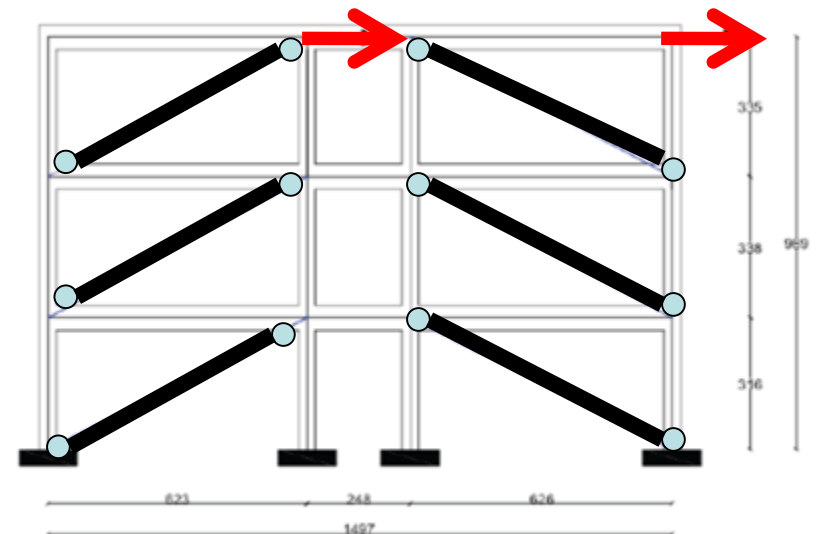
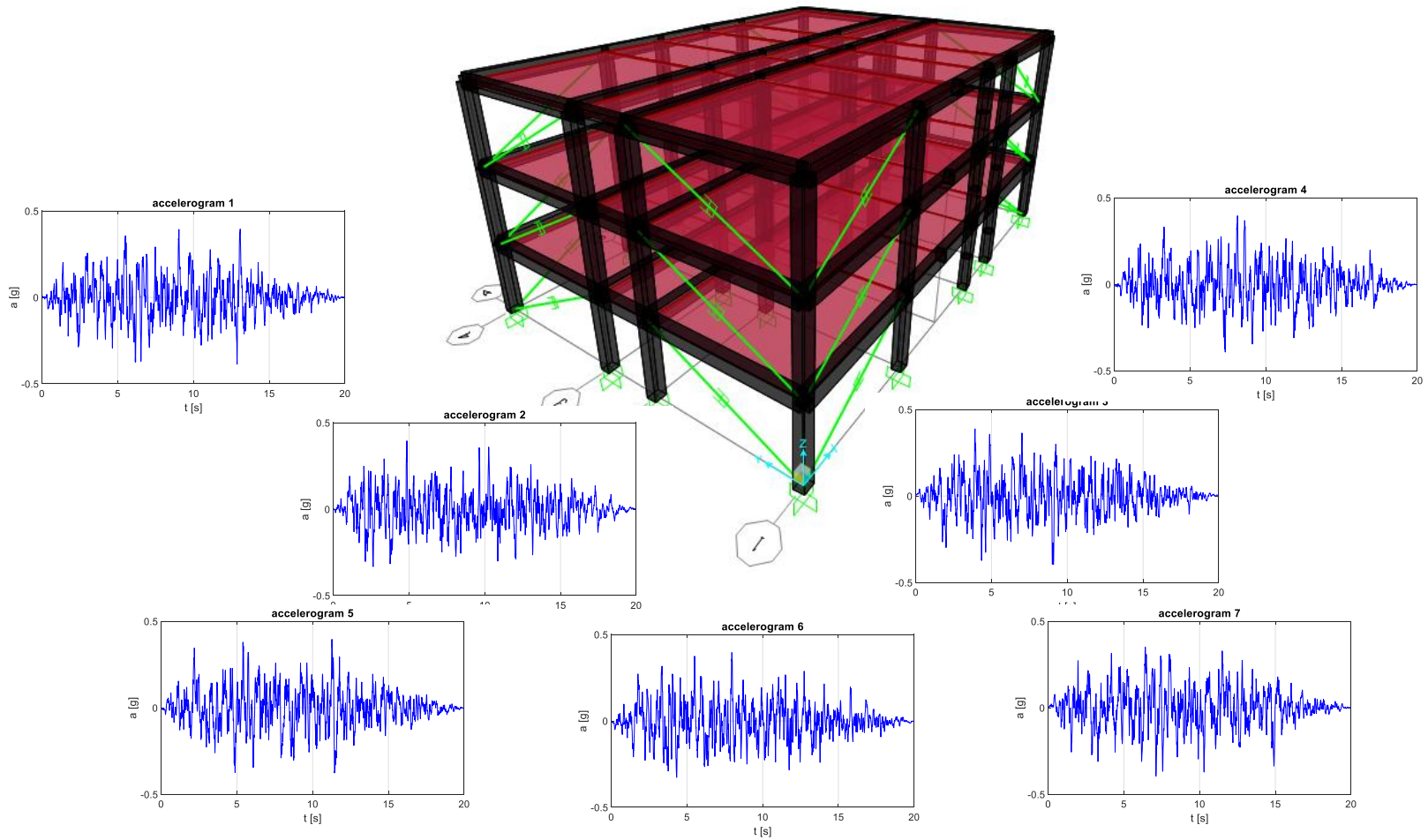


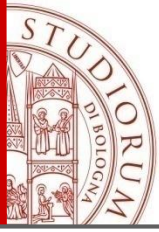
Figure 10. Static scheme to be solved for ESA2.

ESA2

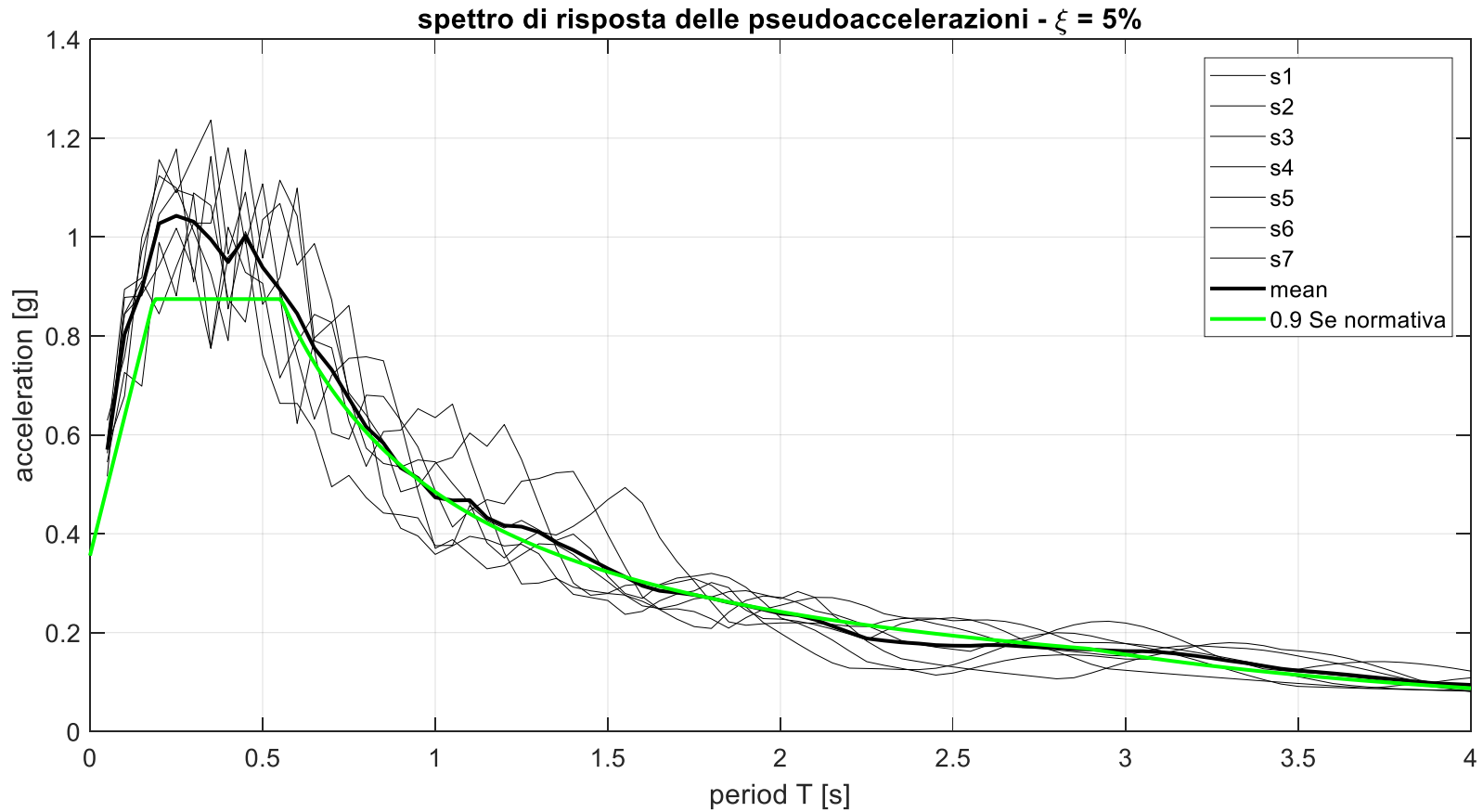
TH verification

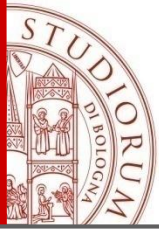
7 accelerograms which are consistent with the design spectrum:





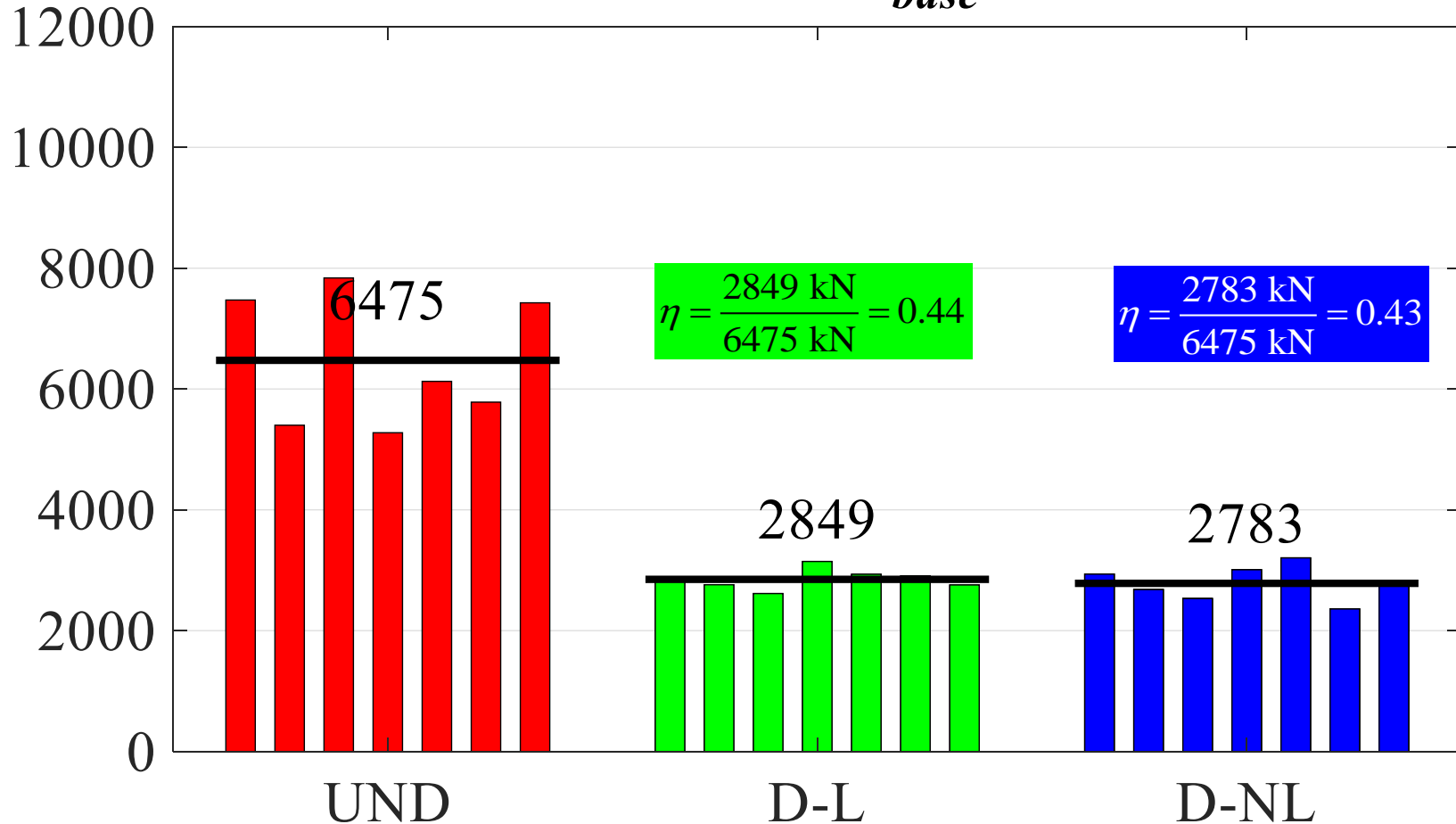
TH verification



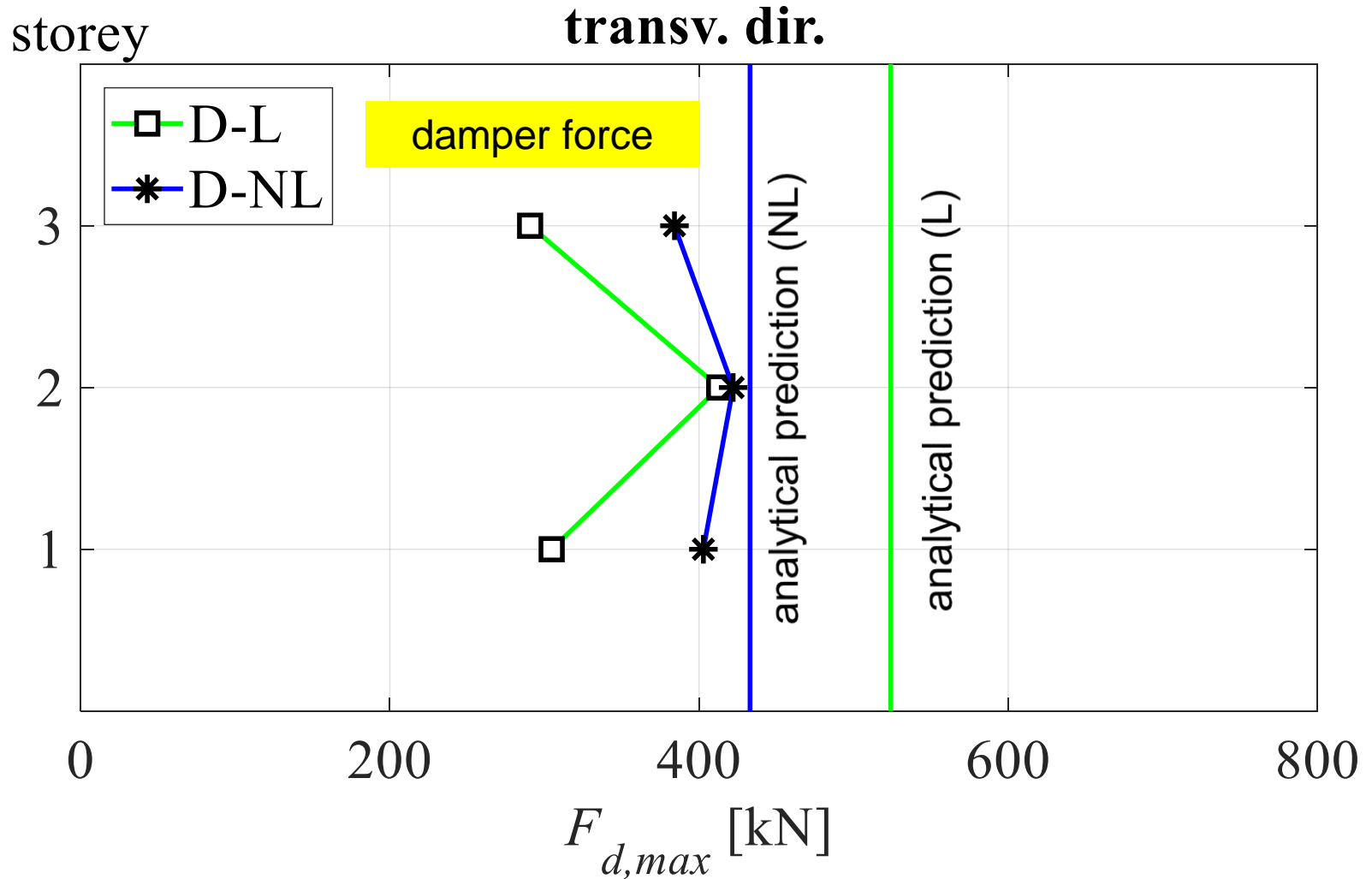


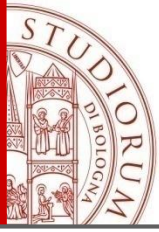
TH verification

transv. dir. V_{base} [kN]

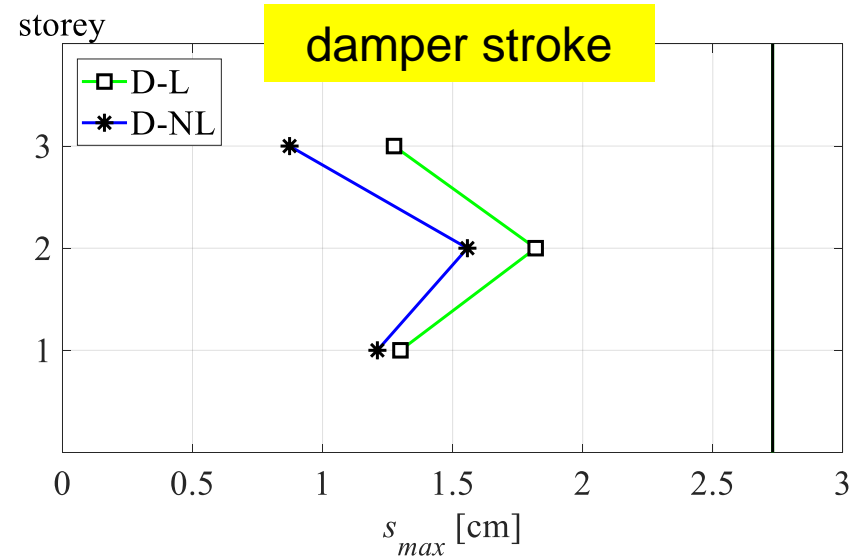
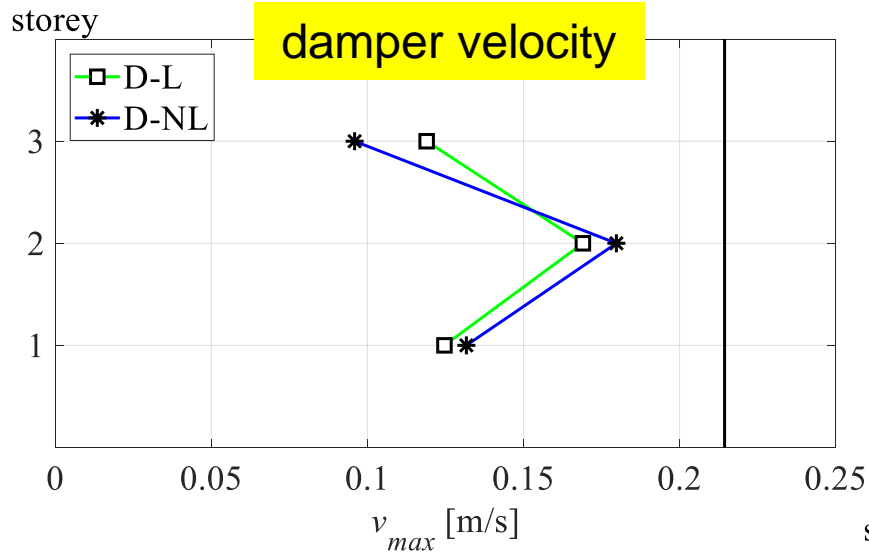


TH verification



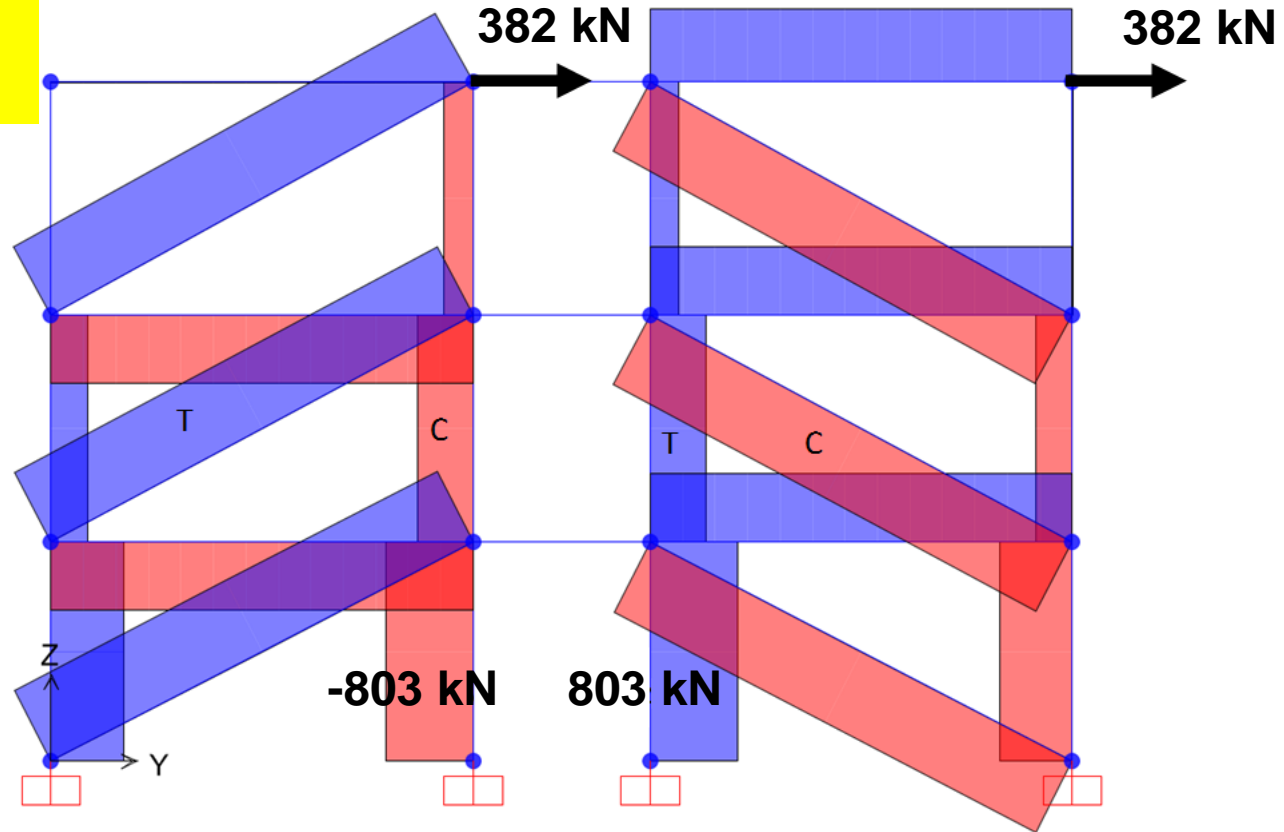


TH verification

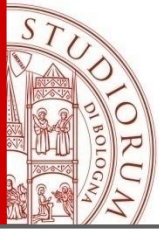


TH verification

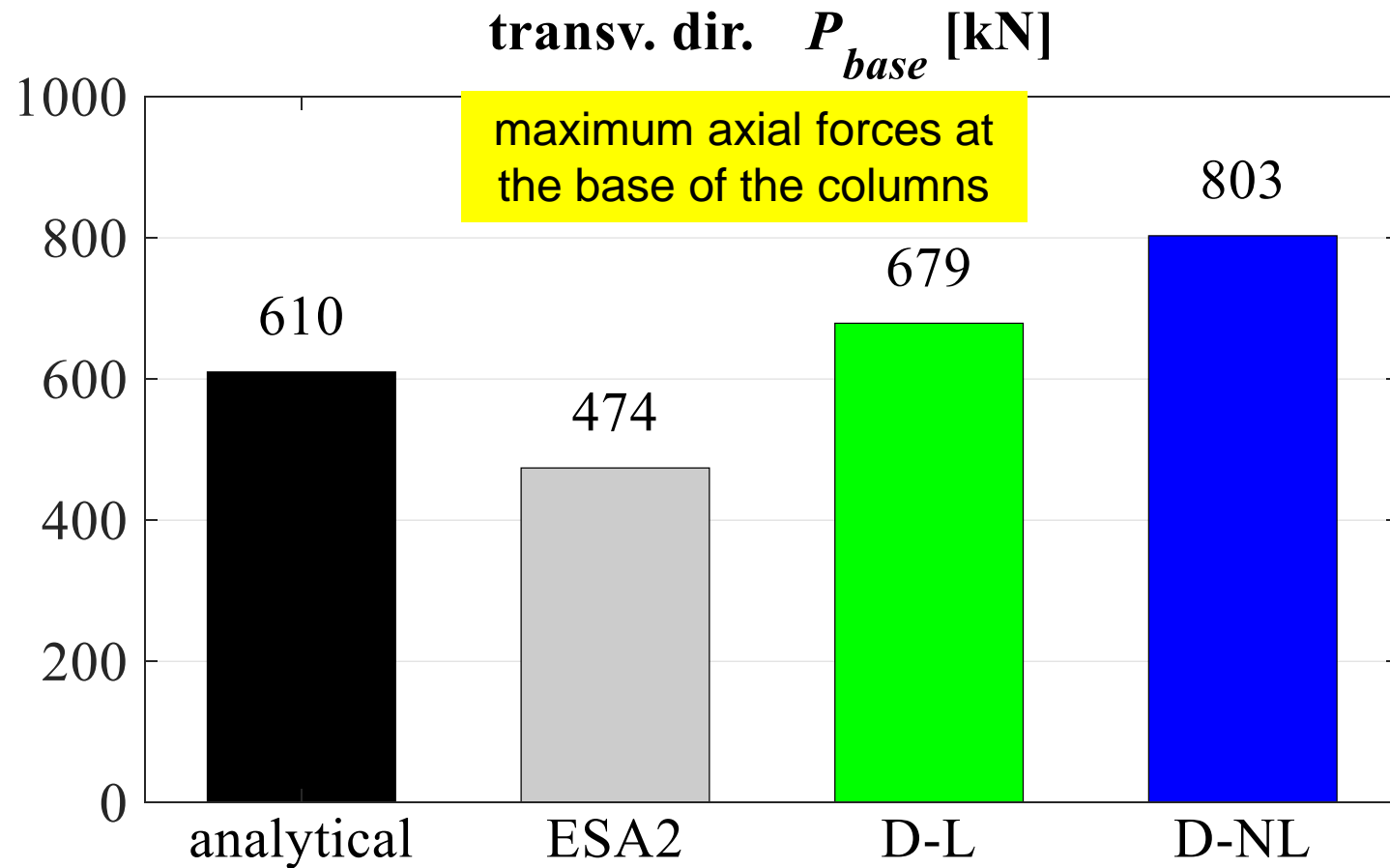
axial forces in the columns

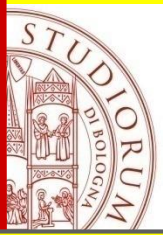


ESA2

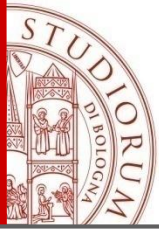


TH verification



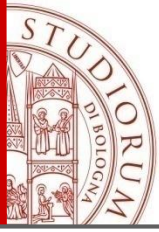


Conclusions



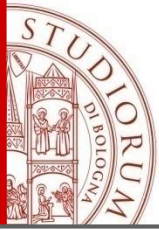
Conclusions

- A **direct (fully analytical)** procedure for the seismic design of building structures with added viscous dampers is presented.
- It represents the **step forward** of the “five-step procedure” (2010).
- It aims at providing **practical tools** for a direct identification of the mechanical characteristics of the manufactured viscous dampers which allow **to achieve target levels of performances**.
- The procedure seems to be **conservative**.
- **In any case, a numerical verification of the final behaviour** of the system by means of non-linear time-history analyses **is recommended**.



Future developments

- In its current version, the procedure is applicable to **regular multi-storey frame structures** which are characterized by a period of vibration lower than 1.5 s.
- At this stage of the research, the procedure is suitable for the **preliminary design phase**, since **correction factors** for the higher modes contributions are necessary to improve its accuracy, especially for high-rise buildings.
- **Other applicative examples are currently under development** for the numerical validation and for the necessary adjustments.



Thank you for your kind attention!

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 - Tomaso Trombetti
 - Giada Gasprini
 - Michele Palermo
 - Luca Landi

Questions?



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